## REVIEW AND SUMMARY FOR CHAPTER 14

## Effective forces

Linear and angular momentum of a system of particles

Motion of the mass center of a system of particles

In this chapter we analyzed the motion of systems of particles, that is, the motion of a large number of particles considered together. In the first part of the chapter we considered systems consisting of well-defined particles, while in the second part we analyzed systems which are continually gaining or losing particles, or doing both at the same time.

We first defined the effective force of a particle $P_{i}$ of a given system as the product $m_{i} \mathbf{a}_{i}$ of its mass $m_{i}$ and its acceleration $\mathbf{a}_{i}$ with respect to a newtonian frame of reference centered at $O$ [Sec. 14.2]. We then showed that the system of the external forces acting on the particles and the system of the effective forces of the particles are equipollent; that is, both systems have the same resultant and the same moment resultant about $O$ :

$$
\begin{align*}
\sum_{i=1}^{n} \mathbf{F}_{i} & =\sum_{i=1}^{n} m_{i} \mathbf{a}_{i}  \tag{14.4}\\
\sum_{i=1}^{n}\left(\mathbf{r}_{i} \times \mathbf{F}_{i}\right) & =\sum_{i=1}^{n}\left(\mathbf{r}_{i} \times m_{i} \mathbf{a}_{i}\right) \tag{14.5}
\end{align*}
$$

Defining the linear momentum $\mathbf{L}$ and the angular momentum $\mathbf{H}_{O}$ about point $O$ of the system of particles [Sec. 14.3] as

$$
\begin{equation*}
\mathbf{L}=\sum_{i=1}^{n} m_{i} \mathbf{v}_{i} \quad \mathbf{H}_{O}=\sum_{i=1}^{n}\left(\mathbf{r}_{i} \times m_{i} \mathbf{v}_{i}\right) \tag{14.6,14.7}
\end{equation*}
$$

we showed that Eqs. (14.4) and (14.5) can be replaced by the equations

$$
\begin{equation*}
\Sigma \mathbf{F}=\dot{\mathbf{L}} \quad \Sigma \mathbf{M}_{O}=\dot{\mathbf{H}}_{O} \tag{14.10,14.11}
\end{equation*}
$$

which express that the resultant and the moment resultant about O of the external forces are, respectively, equal to the rates of change of the linear momentum and of the angular momentum about $O$ of the system of particles.

In Sec. 14.4, we defined the mass center of a system of particles as the point $G$ whose position vector $\overline{\mathbf{r}}$ satisfies the equation

$$
\begin{equation*}
m \overline{\mathbf{r}}=\sum_{i=1}^{n} m_{i} \mathbf{r}_{i} \tag{14.12}
\end{equation*}
$$

where $m$ represents the total mass $\sum_{i=1}^{n} m_{i}$ of the particles. Differentiating both members of Eq. (14.12) twice with respect to $t$, we obtained the relations

$$
\begin{equation*}
\mathbf{L}=m \overline{\mathbf{v}} \quad \dot{\mathbf{L}}=m \overline{\mathbf{a}} \tag{14.14,14.15}
\end{equation*}
$$

where $\overline{\mathbf{v}}$ and $\overline{\mathbf{a}}$ represent, respectively, the velocity and the acceleration of the mass center G. Substituting for $\dot{\mathbf{L}}$ from (14.15) into (14.10), we obtained the equation

$$
\begin{equation*}
\Sigma \mathbf{F}=m \overline{\mathbf{a}} \tag{14.16}
\end{equation*}
$$

from which we concluded that the mass center of a system of particles moves as if the entire mass of the system and all the external forces were concentrated at that point [Sample Prob. 14.1].

In Sec. 14.5 we considered the motion of the particles of a system with respect to a centroidal frame $G x^{\prime} y^{\prime} z^{\prime}$ attached to the mass center $G$ of the system and in translation with respect to the newtonian frame Oxyz (Fig. 14.14). We defined the angular momentum of the system about its mass center $G$ as the sum of the moments about $G$ of the momenta $m_{i} \mathbf{v}_{i}{ }^{\prime}$ of the particles in their motion relative to the frame $G x^{\prime} y^{\prime} z^{\prime}$. We also noted that the same result can be obtained by considering the moments about $G$ of the momenta $m_{i} \mathbf{v}_{i}$ of the particles in their absolute motion. We therefore wrote

$$
\begin{equation*}
\mathbf{H}_{G}=\sum_{i=1}^{n}\left(\mathbf{r}_{i}^{\prime} \times m_{i} \mathbf{v}_{i}\right)=\sum_{i=1}^{n}\left(\mathbf{r}_{i}^{\prime} \times m_{i} \mathbf{v}_{i}^{\prime}\right) \tag{14.24}
\end{equation*}
$$

and derived the relation

$$
\begin{equation*}
\Sigma \mathbf{M}_{G}=\dot{\mathbf{H}}_{G} \tag{14.23}
\end{equation*}
$$

which expresses that the moment resultant about $G$ of the external forces is equal to the rate of change of the angular momentum about $G$ of the system of particles. As will be seen later, this relation is fundamental to the study of the motion of rigid bodies.

When no external force acts on a system of particles [Sec. 14.6], it follows from Eqs. (14.10) and (14.11) that the linear momentum $\mathbf{L}$ and the angular momentum $\mathbf{H}_{O}$ of the system are conserved [Sample Probs. 14.2 and 14.3]. In problems involving central forces, the angular momentum of the system about the center of force $O$ will also be conserved.

The kinetic energy $T$ of a system of particles was defined as the sum of the kinetic energies of the particles [Sec. 14.7]:

$$
\begin{equation*}
T=\frac{1}{2} \sum_{i=1}^{n} m_{i} v_{i}^{2} \tag{14.28}
\end{equation*}
$$

Angular momentum of a system of particles about its mass center


Fig. 14.14

## Conservation of momentum

Principle of work and energy

Conservation of energy

Principle of impulse and momentum

(a)

Fig. 14.15

Use of conservation principles in the solution of problems involving systems of particles

Using the centroidal frame of reference $G x^{\prime} y^{\prime} z^{\prime}$ of Fig. 14.14, we noted that the kinetic energy of the system can also be obtained by adding the kinetic energy $\frac{1}{2} m \bar{v}^{2}$ associated with the motion of the mass center $G$ and the kinetic energy of the system in its motion relative to the frame $G x^{\prime} y^{\prime} z^{\prime}$ :

$$
\begin{equation*}
T=\frac{1}{2} m \bar{v}^{2}+\frac{1}{2} \sum_{i=1}^{n} m_{i} v_{i}^{\prime 2} \tag{14.29}
\end{equation*}
$$

The principle of work and energy can be applied to a system of particles as well as to individual particles [Sec. 14.8]. We wrote

$$
\begin{equation*}
T_{1}+U_{1 \rightarrow 2}=T_{2} \tag{14.30}
\end{equation*}
$$

and noted that $U_{1 \rightarrow 2}$ represents the work of all the forces acting on the particles of the system, internal as well as external.

If all the forces acting on the particles of the system are conservative, we can determine the potential energy $V$ of the system and write

$$
\begin{equation*}
T_{1}+V_{1}=T_{2}+V_{2} \tag{14.31}
\end{equation*}
$$

which expresses the principle of conservation of energy for a system of particles.

We saw in Sec. 14.9 that the principle of impulse and momentum for a system of particles can be expressed graphically as shown in Fig. 14.15. It states that the momenta of the particles at time $t_{1}$ and the impulses of the external forces from $t_{1}$ to $t_{2}$ form a system of vectors equipollent to the system of the momenta of the particles at time $t_{2}$.

(b)

(c)

If no external force acts on the particles of the system, the systems of momenta shown in parts $a$ and $c$ of Fig. 14.15 are equipollent and we have

$$
\begin{equation*}
\mathbf{L}_{1}=\mathbf{L}_{2} \quad\left(\mathbf{H}_{O}\right)_{1}=\left(\mathbf{H}_{O}\right)_{2} \tag{14.36,14.37}
\end{equation*}
$$

Many problems involving the motion of systems of particles can be solved by applying simultaneously the principle of impulse and momentum and the principle of conservation of energy [Sample Prob. 14.4] or by expressing that the linear momentum, angular momentum, and energy of the system are conserved [Sample Prob. 14.5].

In the second part of the chapter, we considered variable systems of particles. First we considered a steady stream of particles, such as a stream of water diverted by a fixed vane or the flow of air through a jet engine [Sec. 14.11]. Applying the principle of impulse and momentum to a system $S$ of particles during a time interval $\Delta t$, and including the particles which enter the system at $A$ during that time interval and those (of the same mass $\Delta m$ ) which leave the system at $B$, we concluded that the system formed by the momentum $(\Delta m) \mathbf{v}_{A}$ of the particles entering $S$ in the time $\Delta t$ and the impulses of the forces exerted on $S$ during that time is equipollent to the momentum $(\Delta m) \mathbf{v}_{B}$ of the particles leaving $S$ in the same time $\Delta t$ (Fig. 14.16). Equating the $x$ components, $y$ components,


Fig. 14.16
and moments about a fixed point of the vectors involved, we could obtain as many as three equations, which could be solved for the desired unknowns [Sample Probs. 14.6 and 14.7]. From this result, we could also derive the following expression for the resultant $\Sigma \mathbf{F}$ of the forces exerted on $S$,

$$
\begin{equation*}
\Sigma \mathbf{F}=\frac{d m}{d t}\left(\mathbf{v}_{B}-\mathbf{v}_{A}\right) \tag{14.39}
\end{equation*}
$$

where $\mathbf{v}_{B}-\mathbf{v}_{A}$ represents the difference between the vectors $\mathbf{v}_{B}$ and $\mathbf{v}_{A}$ and where $d m / d t$ is the mass rate of flow of the stream (see footnote, page 887).

Considering next a system of particles gaining mass by continually absorbing particles or losing mass by continually expelling particles [Sec. 14.12], as in the case of a rocket, we applied the principle of impulse and momentum to the system during a time interval $\Delta t$, being careful to include the particles gained or lost during that time interval [Sample Prob. 14.8]. We also noted that the action on a system $S$ of the particles being absorbed by $S$ was equivalent to a thrust

$$
\begin{equation*}
\mathbf{P}=\frac{d m}{d t} \mathbf{u} \tag{14.44}
\end{equation*}
$$

where $d m / d t$ is the rate at which mass is being absorbed, and $\mathbf{u}$ is the velocity of the particles relative to $S$. In the case of particles being expelled by $S$, the rate $d m / d t$ is negative and the thrust $\mathbf{P}$ is exerted in a direction opposite to that in which the particles are being expelled.

