
Appendix 1

COMPUTER PROGRAM FEM1D

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C      Program Name: FEM1D          Length(INCLUDINMG BLANKS):2440 lines
C
C      * * * * *
C      *              Program FEM1D              *
C      *      (A FINITE ELEMENT ANALYSIS COMPUTER PROGRAM)      *
C      * * * * *
C
C      -----
C      | This is a finite element computer program for the analysis |
C      | of the following three model equations and others:         |
C      |                                                             |
C      | 1. Heat transfer, fluid mechanics, bars, and cables:      |
C      |                                                             |
C      |          CT.u* + CT.u** - (AX.u')' + CX.u = FX           |
C      |                                                             |
C      | 2. The Timoshenko beam and circular plate theory:        |
C      |                                                             |
C      |          CTO.w** - [AX.(w' + s)]' + CX.w = FX           |
C      |          CT1.s** - (BX.s')' + AX.(w' + s) = 0           |
C      |                                                             |
C      | 3. The Euler-Bernoulli beam and circular plate theory:   |
C      |                                                             |
C      |          CT.w** + (BX.w'')'' + CX.w = FX                |
C      |                                                             |
C      | In the above equations (') and (*) denote differentiations |
C      | with respect to space x and time t, and AX, BX, CX, CT, and |
C      | FX are functions of x only:                                |
C      |                                                             |
C      |          AX = AX0 + AX1.X,  BX = BX0 + BX1.X,  CX = CX0 + CX1.X |
C      |          CT = CTO + CT1.X,  FX = FX0 + FX1.X + FX2.X.X     |
C      |                                                             |
C      | In addition to the three model equations, other equations  |
C      | (for example, disks, trusses, and frames) can be analyzed by |
C      | the program.                                               |
C      |-----
C

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C -----
C .
C . KEY VARIABLES USED IN THE PROGRAM .
C . See Table 7.3.2 of the BOOK for a description of the variables. .
C . .
C . NDF..... Number of degrees of freedom per node .
C . NEQ..... Number of equations in the model (before B. C.) .
C . NGP..... Number of Gauss points used in the evaluation of .
C . the element coefficients, ELK , ELF , ELM .
C . NHBW..... Half bandwidth of global coefficient matrix GLK .
C . NN ..... Number of total degrees of freedom in the element .
C . NPE..... Number of nodes per element .
C -----
C .
C . DIMENSIONS OF VARIOUS ARRAYS IN THE PROGRAM .
C . .
C . Values of MXELM,MXNOD, etc. in the PARAMETER statement should .
C . be changed to meet the requirements of the problem: .
C . .
C . MXELM..... Maximum number of elements in the mesh: .
C . MXEBC..... Maximum number of speci. primary deg. of freedom .
C . MXMBC..... Maximum number of speci. mixed boundary conditions .
C . MXNBC..... Maximum number of speci. secondary deg. of freedom .
C . MXNEQ..... Maximum number of equations in the FE model .
C . MXNOD..... Maximum number of nodes in the mesh .
C . .
C . NOTE: The following dimension statement in subroutine JACOBI .
C . should be modified when MXNEQ is greater than 500: .
C . DIMENSION V(500,500),VT(500,500),W(500,500),IH(500) .
C . The value of MXNEQ should be used in place of '500' .
C -----
C .
C . SUBROUTINES USED IN THE PROGRAM .
C . .
C . ASSEMBLE, BOUNDARY, COEFFCNT, CONSTRNT, ECHODATA, EQNSOLVR, .
C . EIGNSLVR, JACOBI, MATRXMLT, MESH1D, POSTPROC, REACTION, .
C . SHAPE1D, TIMFORCE, TIMSTRES, TRANSFRM .
C -----
C
IMPLICIT REAL*8(A-H,O-Z)
PARAMETER (MXELM=250,MXNEQ=500,MXEBC=20,MXNBC=20,MXMBC=20,
* MXNOD=250,MX MPC=5)
DIMENSION DCAX(MXELM,2),DCBX(MXELM,2),DCCX(MXELM,2),DCFX(MXELM,3)
DIMENSION GUO(MXNEQ),GU1(MXNEQ),GU2(MXNEQ),GPU(MXNEQ),DX(MXNOD)
DIMENSION IBDY(MXEBC),ISPV(MXEBC,2),ISSV(MXNBC,2),INBC(MXMBC,2)
DIMENSION IMC1(MX MPC,2),IMC2(MX MPC,2),VMPC(MX MPC,4)
DIMENSION ICON(9),VCON(9),TRM(MXNEQ,MXNEQ)
DIMENSION GLM(MXNEQ,MXNEQ),GLF(MXNEQ),GLX(MXNOD),NOD(MXELM,4)
DIMENSION CS(MXELM),SN(MXELM),CNT(MXELM),SNT(MXELM),XB(MXELM)

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DIMENSION EGNVAL(MXNEQ),EGNVEC(MXNEQ,MXNEQ),GLK(MXNEQ,MXNEQ)
DIMENSION PR(MXELEM),SE(MXELEM),SL(MXELEM),SA(MXELEM),SI(MXELEM)
DIMENSION HF(MXELEM),VF(MXELEM),PF(MXELEM),F3(MXELEM),TITLE(20)
DIMENSION UREF(MXNBC),VSPV(MXEBC),VSSV(MXNBC),VNBC(MXNBC)
COMMON/STF1/ELK(9,9),ELM(9,9),ELF(9),ELX(4),ELU(9),ELV(9),ELA(9)
COMMON/STF2/A1,A2,A3,A4,A5,AX0,AX1,BX0,BX1,CX0,CX1,CT0,CT1,FX0,
*      FX1,FX2
COMMON/IO/IN,IT

C      -----
C      |                                     |
C      |      P R E P R O C E S S O R   U N I T      |
C      |                                     |
C      |-----|
C
IN=5
IT=6
open (in,file=' ')
open (it,file=' ')
NT=0
NSSV=0
JVEC=1
TIME=0.0D0
TOLRNS=1.0D-06
CALL ECHODATA(IN,IT)

C
READ(IN,300) TITLE
READ(IN,*) MODEL,NTYPE,ITEM
READ(IN,*) IELEM,NEM
READ(IN,*) ICONT,NPRNT

C
IF(MODEL.GE.3)THEN
  NPE=2
  IF(MODEL.EQ.4 .AND. NTYPE.GE.1)THEN
    NDF=3
  ELSE
    NDF=2
  ENDIF
  IF(MODEL.EQ.4 .AND. NTYPE.EQ.2)THEN
    IELEM=1
  ELSE
    IELEM=0
  ENDIF
ELSE
  IF(MODEL.EQ.2)THEN
    NDF=2
    IF(NTYPE.GT.1)IELEM=1
  ELSE
    NDF=1
  ENDIF
  NPE=IELEM+1
ENDIF

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C
C Data input for BAR-LIKE and BEAM problems (MODEL=1,2, AND 3)
C
  IF(MODEL.NE.4)THEN
    IF(ICONT.NE.0)THEN
      NNM = NEM*(NPE-1)+1
      NEM1=NEM + 1
      READ(IN,*) (DX(I), I=1,NEM1)
      CALL MESH1D(NEM,NPE,NOD,MXELM,MXNOD,DX,GLX)
      READ(IN,*) AX0,AX1
      READ(IN,*) BX0,BX1
      READ(IN,*) CX0,CX1
      IF(ITEM.LT.3)THEN
        READ(IN,*) FX0,FX1,FX2
      ENDIF
    ELSE
C
C Read GLX, NOD, and element-wise continuous coefficients [DC.X]
C
      READ(IN,*)NNM
      DO 10 N=1,NEM
        READ(IN,*) (NOD(N,I),I=1,NPE), GLX(N)
        READ(IN,*) (DCAX(N,I),I=1,2)
        READ(IN,*) (DCBX(N,I),I=1,2)
        READ(IN,*) (DCCX(N,I),I=1,2)
10      READ(IN,*) (DCFX(N,I),I=1,3)
      ENDIF
    ELSE
C
C Input data for plane TRUSS or FRAME structures (MODEL=4)
C
      READ(IN,*)NNM
      IF(NTYPE.NE.0)THEN
        DO 20 N=1,NEM
          READ(IN,*) PR(N),SE(N),SL(N),SA(N),SI(N),CS(N),SN(N)
          READ(IN,*) HF(N),VF(N),PF(N),XB(N),CNT(N),SNT(N)
20      READ(IN,*) (NOD(N,I),I=1,2)
        ELSE
          DO 30 N=1,NEM
            READ(IN,*) SE(N),SL(N),SA(N),CS(N),SN(N),HF(N)
30      READ(IN,*) (NOD(N,I),I=1,2)
          ENDIF
          READ(IN,*) NCON
          IF(NCON.NE.0)THEN
            DO 35 I=1, NCON
35      READ(IN,*) ICON(I),VCON(I)
          ENDIF
        ENDIF
      ENDIF
      NEQ=NNM*NDF
C

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C   Read data on BOUNDARY CONDITIONS of three kinds: Dirichlet (PV)
C       Neumann (SV), and Newton's (MIXED) types
C
      READ(IN,*) NSPV
      IF(NSPV.NE.0) THEN
        DO 40 NB=1,NSPV
          IF(ITEM.GT.2) THEN
            READ(IN,*) (ISPV(NB,J),J=1,2)
          ELSE
            READ(IN,*) (ISPV(NB,J),J=1,2),VSPV(NB)
          ENDIF
40      CONTINUE
      ENDIF
C
      IF(ITEM.LE.2) THEN
        READ(IN,*) NSSV
        IF(NSSV.NE.0) THEN
          DO 50 IB=1,NSSV
50      READ(IN,*) (ISSV(IB,J),J=1,2),VSSV(IB)
          ENDIF
        ENDIF
C
        READ(IN,*) NNBC
        IF(NNBC.NE.0) THEN
          DO 60 I=1, NNBC
60      READ(IN,*) (INBC(I,J),J=1,2),VNBC(I),UREF(I)
          ENDIF
C
C   Read data on multi-point constraints
C
      READ(IN,*) NMPC
      IF(NMPC.NE.0) THEN
        DO 65 I=1, NMPC
65      READ(IN,*) (IMC1(I,J),J=1,2), (IMC2(I,J),J=1,2), (VMPC(I,J),J=1,4)
          ENDIF
C
      IF(ITEM.NE.0) THEN
C
C   Input data here for TIME-DEPENDENT problems
C
        IF(ITEM.LE.3) THEN
          READ(IN,*) CT0,CT1
        ENDIF
        IF(ITEM.LE.2) THEN
          READ(IN,*) DT,ALFA,GAMA
          READ(IN,*) INCOND,NTIME,INTVL
          A1=ALFA*DT
          A2=(1.0-ALFA)*DT
          IF(INCOND.NE.0) THEN
            READ(IN,*) (GUO(I),I=1,NEQ)
          ENDIF
        ENDIF
      ENDIF

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ELSE
  DO 70 I=1,NEQ
70    GUO(I)=0.0
  ENDF
  IF (ITEM.EQ.2) THEN
    A3=2.0/GAMA/(DT*DT)
    A4=A3*DT
    A5=1.0/GAMA-1.0
    IF (INCOND.NE.0) THEN
      READ(IN,*) (GU1(I),I=1,NEQ)
    ELSE
      DO 80 I=1,NEQ
80    GU1(I)=0.0
      GU2(I)=0.0
    ENDF
  ENDF
ENDF
ENDF
ENDF
C
C -----
C   E   N   D   O   F   T   H   E   I   N   P   U   T   D   A   T   A
C -----
C
C   Compute the half BANDWIDTH of the coefficient matrix GLK
C
  NHBW=0.0
  DO 90 N=1,NEM
  DO 90 I=1,NPE
  DO 90 J=1,NPE
    NW=(IABS(NOD(N,I)-NOD(N,J))+1)*NDF
90 IF(NHBW.LT.NW) NHBW=NW
C
C -----
C                   P R I N T   T H E   I N P U T   D A T A
C -----
C
  WRITE(IT,530)
  WRITE(IT,310)
  WRITE(IT,530)
  WRITE(IT,300) TITLE
  WRITE(IT,320) MODEL,NTYPE
  WRITE(IT,350) IELEM,NDF,NEM,NEQ,NHBW,NSPV,NSSV,NNBC,NMPC
C
  IF (ITEM.NE.0) THEN
    IF (ITEM.LE.2) THEN
      WRITE(IT,330)
      WRITE(IT,390) CTO,CT1,ALFA,GAMA,DT,NTIME,INTVL
      IF (INCOND.NE.0) THEN
        WRITE(IT,370)
        WRITE(IT,540) (GUO(I),I=1,NEQ)
      
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        IF (ITEM.EQ.2) THEN
            WRITE(IT,380)
            WRITE(IT,540) (GU1(I),I=1,NEQ)
        ENDIF
    ENDIF
ELSE
    WRITE(IT,340)
    IF (ITEM.LE.3) THEN
        WRITE(IT,400) CTO,CT1
    ENDIF
ENDIF
ENDIF
ENDIF
C
IF (NSPV.NE.0) THEN
    WRITE(IT,480)
    DO 100 IB=1,NSPV
        IF (ITEM.LE.2) THEN
            WRITE(IT,490) (ISPV(IB,J),J=1,2),VSPV(IB)
        ELSE
            WRITE(IT,490) (ISPV(IB,J),J=1,2)
        ENDIF
    100 CONTINUE
ENDIF
C
IF (NSSV.NE.0) THEN
    WRITE(IT,500)
    DO 110 IB=1,NSSV
    110 WRITE(IT,490) (ISSV(IB,J),J=1,2),VSSV(IB)
    ENDIF
C
IF (NNBC.NE.0) THEN
    WRITE(IT,510)
    DO 120 I=1,NNBC
    120 WRITE(IT,490) (INBC(I,J),J=1,2),VNBC(I),UREF(I)
    ENDIF
C
IF (NMPC.NE.0) THEN
    WRITE(IT,515)
    DO 125 I=1, NMPC
    125 WRITE(IT,495) (IMC1(I,J),J=1,2), (IMC2(I,J),J=1,2),
        * (VMPC(I,J),J=1,4)
    ENDIF

IF (MODEL.NE.4) THEN
    IF (ICONT.EQ.1) THEN
        WRITE(IT,410)
        WRITE(IT,540) (GLX(I),I=1,NNM)
        WRITE(IT,420)
        IF (MODEL.NE.3) THEN
            WRITE(IT,440) AX0,AX1,BX0,BX1,CX0,CX1,FX0,FX1,FX2

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      ELSE
        WRITE(IT,445) AX0,AX1,BX0,BX1,CX0,CX1
      ENDIF
    ELSE
      DO 130 N=1,NEM
        WRITE(IT,430) N, GLX(N)
130      WRITE(IT,440) (DCAX(N,I), I=1,2), (DCBX(N,I), I=1,2),
      *              (DCCX(N,I), I=1,2), (DCFX(N,I), I=1,3)
      ENDIF
    ELSE
      DO 140 N=1,NEM
        WRITE(IT,460) N
        IF(NTYPE.NE.0) THEN
          WRITE(IT,450) PR(N),SE(N),SL(N),SA(N),SI(N),CS(N),SN(N),
          *              HF(N),VF(N),PF(N),XB(N),CNT(N),SNT(N),
          *              (NOD(N,I), I=1,2)
        ELSE
          WRITE(IT,470) SE(N),SL(N),SA(N),CS(N),SN(N),HF(N),
          *              (NOD(N,I), I=1,2)
        ENDIF
140      CONTINUE
      ENDIF
C
C      -----
C      |               P R O C E S S O R   U N I T               |
C      |-----|
C
C      TIME MARCHING scheme begins here. For ITEM=2, initial conditions
C      on second derivatives of the solution are computed in the program
C
      IF(ITEM.NE.0) THEN
        IF(ITEM.EQ.1) THEN
          NT=NT+1
          TIME=TIME+DT
        ENDIF
      ENDIF
C
      IF(ITEM.GE.3) NHBW=NEQ
C
C      Initialize global matrices and vectors
C
150 DO 160 I=1,NEQ
      GLF(I)=0.0
      DO 160 J=1,NHBW
        IF(ITEM.GE.3) THEN
          GLM(I,J)=0.0
        ENDIF
160 GLK(I,J)=0.0

```



```

C
C Do-loop for ELEMENT CALCULATIONS and ASSEMBLY
C
DO 200 NE = 1, NEM
IF(MODEL.NE.4)THEN
  IF(ICONT.NE.1) THEN
    AXO=DCA(X,NE,1)
    AX1=DCA(X,NE,2)
    BXO=DCB(X,NE,1)
    BX1=DCB(X,NE,2)
    CXO=DCC(X,NE,1)
    CX1=DCC(X,NE,2)
    FXO=DCF(X,NE,1)
    FX1=DCF(X,NE,2)
    FX2=DCF(X,NE,3)
  ENDIF
C
  L=0
  DO 180 I=1,NPE
    NI=NOD(NE,I)
    IF(ICONT.EQ.1)THEN
      ELX(I)=GLX(NI)
    ELSE
      ELX(1)=0.0
      ELX(2)=0.5*GLX(NE)
      ELX(NPE)=GLX(NE)
    ENDIF
    IF(ITEM.EQ.1 .OR. ITEM.EQ.2)THEN
      LI=(NI-1)*NDF
      DO 170 J=1,NDF
        LI=LI+1
        L=L+1
        ELU(L)=GUO(LI)
        IF(ITEM.EQ.2 .AND. NT.GT.0)THEN
          ELV(L)=GU1(LI)
          ELA(L)=GU2(LI)
        ENDIF
170      CONTINUE
      ENDIF
180      CONTINUE
C
      CALL COEFFCNT(IELEM,ITEM,MODEL,NDF,NPE,TIME,NTYPE,NE,F3,MXELM)
    ELSE
      CALL TRANSFRM(MXELM,NE,NTYPE,PR,SE,SL,SA,SI,CS,SN,CNT,SNT,
*                HF,VF,PF,XB)
    ENDIF

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C
  IF(NPRNT .NE.0) THEN
    NN = NPE*NDF
    IF(NPRNT .LE.2) THEN
      IF(NE.LE.5 .AND. NT.LE.1) THEN
        WRITE(IT,550)
        DO 190 I=1,NN
190      WRITE(IT,540) (ELK(I,J),J=1,NN)
          IF(ITEM.GE.3) THEN
            WRITE(IT,360)
            DO 195 I=1,NN
195      WRITE(IT,540) (ELM(I,J),J=1,NN)
          ELSE
            WRITE(IT,560)
            WRITE(IT,540) (ELF(I),I=1,NN)
          ENDIF
        ENDIF
      ENDIF
    ENDIF
  ENDIF
C
C Assemble element matrices
C
  CALL ASSEMBLE(NOD,MXELM,MXNEQ,NDF,NPE,NE,ITEM,GLK,GLM,GLF)
C
200 CONTINUE
C
C Call subroutine CONSTRNT to impose constraint boundary conditions,
C for example, inclined support conditions
C
  IF(MODEL.EQ.4) THEN
    IF(NCON.NE.0) THEN
      CALL CONSTRNT(NEQ,NHBW,NDF,NCON,ICON,VCON,GLK,GLM,GLF,TRM,MXNEQ)
    ENDIF
  ENDIF
C
C Impose multi-point constraints using the penalty method
C
  IF(NMPC.NE.0) THEN
    IF(NPRNT.EQ.2) THEN
      WRITE(IT,570)
      DO 201 I=1,NEQ
201    WRITE(IT,540) (GLK(I,J),J=1,NHBW)
    ENDIF
    VMAX=0.0
    DO 204 I=1,NEQ
      DO 204 J=I,NHBW
        VALUE=DABS(GLK(I,J))
        IF(VALUE.GT.VMAX) THEN
          VMAX=VALUE
        ENDIF
      ENDIF
    ENDIF
  ENDIF

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204  CONTINUE
      PNLTY=VMAX*1.OE4
      DO 205 NC=1,NMPC
        NDOF1=(IMC1(NC,1)-1)*NDF+IMC1(NC,2)
        NDOF2=(IMC2(NC,1)-1)*NDF+IMC2(NC,2)
        GLK(NDOF1,1)=GLK(NDOF1,1)+PNLTY*VMPC(NC,1)*VMPC(NC,1)
        GLK(NDOF2,1)=GLK(NDOF2,1)+PNLTY*VMPC(NC,2)*VMPC(NC,2)
        GLF(NDOF1)=GLF(NDOF1)+PNLTY*VMPC(NC,1)*VMPC(NC,3)
        GLF(NDOF2)=GLF(NDOF2)+PNLTY*VMPC(NC,2)*VMPC(NC,3)
        IF(NDOF1.GT.NDOF2)THEN
          NW=NDOF1-NDOF2+1
          GLK(NDOF2,NW)=GLK(NDOF2,NW)+PNLTY*VMPC(NC,1)*VMPC(NC,2)
          GLF(NDOF1)=VMPC(NC,4)
        ELSE
          NW=NDOF2-NDOF1+1
          GLK(NDOF1,NW)=GLK(NDOF1,NW)+PNLTY*VMPC(NC,1)*VMPC(NC,2)
          GLF(NDOF2)=VMPC(NC,4)
        ENDIF
205  CONTINUE
      ENDIF
C
      IF(NPRNT.EQ.2)THEN
C
C    Print assembled coefficient matrices if required
C
      WRITE(IT,570)
      DO 210 I=1,NEQ
210  WRITE(IT,540) (GLK(I,J),J=1,NHBW)
      IF(ITEM.GE.3)THEN
        WRITE(IT,575)
        DO 215 I=1,NEQ
215  WRITE(IT,540) (GLM(I,J),J=1,NHBW)
      ELSE
        WRITE(IT,580)
        WRITE(IT,540) (GLF(I),I=1,NEQ)
      ENDIF
      ENDIF
C
C    Call subroutine BOUNDARY to impose essential, natural and Newton's
C    type boundary conditions on the primary and secondary variables.
C
      CALL BOUNDARY(NEQ,NEQR,NHBW,NSPV,NSSV,NNBC,NDF,DT,ITEM,ALFA,IBDY,
*                ISPV,ISSV,INBC,UREF,VSPV,VSSV,VNBC,GLK,GLM,GLF,GUO,
*                MXEBC,MXNBC,MXNBC,MXNEQ)

      IF(NPRNT.EQ.2)THEN
C
C    Print assembled coefficient matrices if required
C

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        WRITE(IT,570)
        DO 211 I=1,NEQ
211    WRITE(IT,540) (GLK(I,J),J=1,NHBW)
        ENDIF
C
        IF(ITEM.GE.3)THEN
C
C    Call EGN SOLVR to solve for the eigenvalues and eigenvectors
C
        CALL EGN SOLVR(NEQR,GLK,GLM,EGNVAL,EGNVEC,JVEC,NROT,MXNEQ)
C
        WRITE(IT,690) NROT
        DO 230 NVEC=1,NEQR
        FRQNCY=DSQRT(EGNVAL(NVEC))
        WRITE(IT,700)NVEC,EGNVAL(NVEC),FRQNCY
230    WRITE(IT,540) (EGNVEC(I,NVEC),I=1,NEQR)
        STOP
        ENDIF
C
        IRES = 0
C
C    Call subroutine EQNSOLVR to solve the finite-element equations
C
        CALL EQNSOLVR(MXNEQ,MXNEQ,NEQ,NHBW,GLK,GLF,IRES)
C
        IF(ITEM.EQ.0)THEN
        WRITE(IT,590)
        WRITE(IT,540) (GLF(NI),NI=1,NEQ)
        ELSE
        IF(NT.EQ.0)THEN
        DO 240 I=1,NEQ
240    GU2(I)=GLF(I)
        NT=NT+1
        TIME=TIME+DT
        GOTO 150
        ENDIF
C
C    Compute and print current values of GU0, GU1, and GU2
C
        DO 250 I=1,NEQ
        IF(ITEM.EQ.2)THEN
        ACCLRN=A3*(GLF(I)-GU0(I))-A4*GU1(I)-A5*GU2(I)
        GU1(I)=GU1(I)+A2*GU2(I)+A1*ACCLRN
        GU2(I)=ACCLRN
        GPU(I)=GU0(I)
        ELSE
        GPU(I)=GU0(I)
        ENDIF
250    GU0(I)=GLF(I)
C

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```

DIFF=0.0
SOLN=0.0
DO 260 I=1,NEQ
SOLN=SOLN+GU0(I)*GU0(I)
260 DIFF=DIFF+(GLF(I)-GPU(I))**2
PRCNT=DSQRT(DIFF/SOLN)
IF (PRCNT.LE.TOLRNS) THEN
WRITE(IT,640)
WRITE(IT,540) (GPU(I),I=1,NEQ)
WRITE(IT,540) (GU0(I),I=1,NEQ)
STOP
ELSE
IF (INTVL.LE.0) INTVL=1
NTEN=(NT/INTVL)*INTVL
IF (NTEN.EQ.NT) THEN
WRITE(IT,600) TIME, NT
WRITE(IT,590)
WRITE(IT,540) (GU0(I),I=1,NEQ)
IF (ITEM.NE.1) THEN
IF (NPRNT.LT.4) THEN
WRITE(IT,645)
WRITE(IT,540) (GU1(I),I=1,NEQ)
WRITE(IT,646)
WRITE(IT,540) (GU2(I),I=1,NEQ)
ENDIF
ENDIF
NT=NT+1
TIME=TIME+DT
ELSE
NT=NT+1
TIME=TIME+DT
GOTO 150
ENDIF
ENDIF
ENDIF
ENDIF
IF (NMPC.EQ.0) THEN
IF (NPRNT.LE.1) THEN
IF (MODEL.EQ.1) THEN
WRITE(IT,530)
ELSE
IF (MODEL.EQ.4) THEN
WRITE(IT,630)
ENDIF
WRITE(IT,520)
ENDIF
ENDIF
C
IF (MODEL.EQ.1) THEN
WRITE(IT,647)
IF (NTYPE.EQ.0) THEN

```

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```

        WRITE(IT, 610)
        ELSE
            WRITE(IT, 620)
        ENDIF
    C
    ENDIF
    IF (MODEL.EQ.2 .OR. MODEL.EQ.3) THEN
        WRITE(IT, 647)
        IF (NTYPE.EQ.0) THEN
            WRITE(IT, 650)
        ELSE
            WRITE(IT, 660)
        ENDIF
    C
    ENDIF
    IF (MODEL.EQ.4) THEN
        IF (NTYPE.EQ.0) THEN
            WRITE(IT, 680)
        ELSE
            WRITE(IT, 670)
        ENDIF
    C
    ENDIF
    IF (MODEL.EQ.1) THEN
        WRITE(IT, 530)
    ELSE
        WRITE(IT, 520)
    ENDIF
    C
    IF (MODEL.LE.3) THEN
        CALL POSTPROC(DCAX, DCBX, DCCX, F3, GLF, GLX, NOD, ICONT, IELEM, NPE,
*           MODEL, NTYPE, ITEM, MXELM, MXNEQ, MXNOD, NEM, NDF)
    ELSE
        CALL REACTION(MXELM, MXNEQ, NDF, NEM, NOD, NPE, NTYPE, PR, GLF,
*           SE, SL, SA, SI, CS, SN, CNT, SNT, HF, VF, PF, XB)
    ENDIF
    C
    IF (MODEL.EQ.1) THEN
        WRITE(IT, 530)
    ELSE
        WRITE(IT, 520)
    ENDIF
    ENDIF
    ELSE
    C
    C Calculate the reactions at the points where constraints are imposed
    C
        DO 280 NC=1, NMPC
            NDOF1=(IMC1(NC,1)-1)*NDF+IMC1(NC,2)
            NDOF2=(IMC2(NC,1)-1)*NDF+IMC2(NC,2)

```

```

      GUO(NC)=-PNLTY*VMPC(NC,1)*(VMPC(NC,1)*GLF(NDOF1)
*          +VMPC(NC,2)*GLF(NDOF2)-VMPC(NC,3))
      GU1(NC)=-PNLTY*VMPC(NC,2)*(VMPC(NC,1)*GLF(NDOF1)
*          +VMPC(NC,2)*GLF(NDOF2)-VMPC(NC,3))
280  CONTINUE
      WRITE(IT,545)
      WRITE(IT,540)(GUO(I),I=1,NMPC)
      WRITE(IT,540)(GU1(I),I=1,NMPC)
      ENDIF

      IF(ITEM.EQ.0)STOP
      IF(NT.LT.NTIME)THEN
        IF(PRCNT.GT.TOLRNS)THEN
          GOTO 150
        ENDIF
      ELSE
        WRITE(IT,710)
      ENDIF

C -----
C           F O R M A T S
C -----
300 FORMAT(20A4)
310 FORMAT(8X,'OUTPUT from program  FEM1D  by J N REDDY')
320 FORMAT(/,4X,'*** ANALYSIS OF MODEL',I2,', AND TYPE',I2,
*          ' PROBLEM ***',/,15X,'(see the code below)',/,
*          /,4X,'MODEL=1,NTYPE=0: A problem described by MODEL EQ. 1',
*          /,4X,'MODEL=1,NTYPE=1: A circular DISK (PLANE STRESS) ',
*          /,4X,'MODEL=1,NTYPE>1: A circular DISK (PLANE STRAIN) ',
*          /,4X,'MODEL=2,NTYPE=0: A Timoshenko BEAM (RIE) problem',
*          /,4X,'MODEL=2,NTYPE=1: A Timoshenko PLATE (RIE) problem',
*          /,4X,'MODEL=2,NTYPE=2: A Timoshenko BEAM (CIE) problem',
*          /,4X,'MODEL=2,NTYPE>2: A Timoshenko PLATE (CIE) problem',
*          /,4X,'MODEL=3,NTYPE=0: A Euler-Bernoulli BEAM problem',
*          /,4X,'MODEL=3,NTYPE>0: A Euler-Bernoulli Circular plate',
*          /,4X,'MODEL=4,NTYPE=0: A plane TRUSS problem',
*          /,4X,'MODEL=4,NTYPE=1: A Euler-Bernoulli FRAME problem',
*          /,4X,'MODEL=4,NTYPE=2: A Timoshenko (CIE) FRAME problem',/)
330 FORMAT(/,4X,'TIME-DEPENDENT (TRANSIENT) ANALYSIS ',/)
340 FORMAT(/,4X,'E I G E N V A L U E  A N A L Y S I S',/)
350 FORMAT(/,8X,'Element type (0, Hermite,>0, Lagrange)..=',I4,/,
*          8X,'No. of deg. of freedom per node, NDF....=',I4,/,
*          8X,'No. of elements in the mesh, NEM.....=',I4,/,
*          8X,'No. of total DOF in the model, NEQ.....=',I4,/,
*          8X,'Half bandwidth of matrix [GLK], NHBW ...=',I4,/,
*          8X,'No. of specified primary DOF, NSPV.....=',I4,/,
*          8X,'No. of specified secondary DOF, NSSV....=',I4,/,
*          8X,'No. of specified Newton B. C.: NNBC.....=',I4,/,
*          8X,'No. of speci. multi-pt. cond.: NMPC.....=',I4)
360 FORMAT(/,3X,'Element coefficient matrix, [ELM]:',/)
370 FORMAT(/,3X,'Initial conditions on the primary variables:',/)

```

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```

380 FORMAT(/,3X, 'Initial cond. on time der. of primary variables:',/)
390 FORMAT(/,8X,'Coefficient, CT0.....=' ,E12.4,/,
*      8X,'Coefficient, CT1.....=' ,E12.4,/,
*      8X,'Parameter, ALFA.....=' ,E12.4,/,
*      8X,'Parameter, GAMA.....=' ,E12.4,/,
*      8X,'Time increment, DT.....=' ,E12.4,/,
*      8X,'No. of time steps, NTIME.....=' ,I4,/,
*      8X,'Time-step interval to print soln., INTVL=' ,I4,/)
400 FORMAT(/,8X,'Coefficient, CT0.....=' ,E12.4,/,
*      8X,'Coefficient, CT1.....=' ,E12.4,/)
410 FORMAT(/,3X,'Global coordinates of the nodes, {GLX}:' ,/)
420 FORMAT(/,3X,'Coefficients of the differential equation:' ,/)
430 FORMAT(/,5X,'Properties of Element =' ,I3,/,
*      8X,'Element length, H ..... =' ,E12.4)
440 FORMAT( 8X,'AX0 =' ,E12.4,5X,'AX1 =' ,E12.4,/,
*      8X,'BX0 =' ,E12.4,5X,'BX1 =' ,E12.4,/,
*      8X,'CX0 =' ,E12.4,5X,'CX1 =' ,E12.4,/,
*      8X,'FX0 =' ,E12.4,5X,'FX1 =' ,E12.4,5X,'FX2 =' ,E12.4,/)
445 FORMAT( 8X,'AX0 =' ,E12.4,5X,'AX1 =' ,E12.4,/,
*      8X,'BX0 =' ,E12.4,5X,'BX1 =' ,E12.4,/,
*      8X,'CX0 =' ,E12.4,5X,'CX1 =' ,E12.4,/)
450 FORMAT(8X,'The poisson ratio,          PR..... =' ,E12.4,/,
*      8X,'Modulus of elasticity,        SE..... =' ,E12.4,/,
*      8X,'Length of the element,        SL..... =' ,E12.4,/,
*      8X,'Area of cross section,        SA..... =' ,E12.4,/,
*      8X,'Moment of inertia,            SI..... =' ,E12.4,/,
*      8X,'Cosine of orientation,        CN..... =' ,E12.4,/,
*      8X,'Sine of orientation,          SN..... =' ,E12.4,/,
*      8X,'Axial body force (constant),  HF..... =' ,E12.4,/,
*      8X,'Transverse body force (cnst), VF..... =' ,E12.4,/,
*      8X,'Internal point force,         PF..... =' ,E12.4,/,
*      8X,'Location of PF from node 1,   XB..... =' ,E12.4,/,
*      8X,'Orientation of PF: cosine,    CST..... =' ,E12.4,/,
*      8X,'Orientation of PF: sine,     SNT..... =' ,E12.4,/,
*      8X,'Nodal connectivity:          NOD(I,J).. =' ,2I6,/)

460 FORMAT(/,3X,'Element No. =' , I3,/)
470 FORMAT(8X,'Modulus of elasticity,        SE..... =' ,E12.4,/,
*      8X,'Length of the element,        SL..... =' ,E12.4,/,
*      8X,'Area of cross section,        SA..... =' ,E12.4,/,
*      8X,'Cosine of orientation,        CN..... =' ,E12.4,/,
*      8X,'Sine of orientation,          SN..... =' ,E12.4,/,
*      8X,'Axial body force (constant),  HF..... =' ,E12.4,/,
*      8X,'Nodal connectivity:          NOD(I,J).. =' ,2I6,/)
480 FORMAT(/,3X, 'Boundary information on primary variables:' ,/)
490 FORMAT(5X,2I5,2E15.5)
495 FORMAT(5X,2I5,2X,2I5,/,5X,4E15.5)
500 FORMAT(/,3X, 'Boundary information on secondary variables:' ,/)
510 FORMAT(/,3X, 'Boundary information on mixed boundary cond.:' ,/)
515 FORMAT(/,3X, 'Multi-point constraint information:' ,/)

```



```

520 FORMAT(2X,78('_ '),/)
530 FORMAT(2X,55('_ '),/)
540 FORMAT(2X,5E13.5)
545 FORMAT(/,3X,'Forces at the constrained points:',/)
550 FORMAT(/,3X,'Element coefficient matrix, [ELK]:',/)
560 FORMAT(/,3X,'Element source vector, {ELF}:',/)
570 FORMAT(/,3X,'Global coefficient matrix, [GLK]:',/)
575 FORMAT(/,3X,'Global coefficient matrix, [GLM]:',/)
580 FORMAT(/,3X,'Global source vector, {GLF}:',/)
590 FORMAT(/,1X,'SOLUTION (values of PVs) at the NODES: ',/)
600 FORMAT(/,1X,'TIME =',E12.4,5X,'Time step number =',I3,/)
610 FORMAT(7X,' x ',5X,'P. Variable',2X,'S. Variable')
620 FORMAT(7X,' x ',5X,'Displacement',2X,'Radial Stress',2X,
*      'Hoop Stress')
630 FORMAT(/,9X,'Generalized forces in the element coordinates',/,
*      5X,'(second line gives the results in the global coordinates)')
640 FORMAT(/,3X,'*** THE SOLUTION HAS REACHED A STEADY STATE ***',
*      /,3X,'SOLUTION AT THE TWO CONSECUTIVE TIME STEPS FOLLOWS:')
645 FORMAT(/,2X,'FIRST TIME DERIVATIVE of the primary variables:',/)
646 FORMAT(/,2X,'SECOND TIME DERIVATIVE of the primary variables:',/)
647 FORMAT(3X,'x is the global coord. if ICONT=1 and it is the local',
*      ' coord. if ICONT=0')
650 FORMAT(7X,' x ',6X,'Deflect.',5X,'Rotation',5X,'B. Moment',
*      3X,'Shear Force')
660 FORMAT(7X,' x ',6X,'Deflect.',5X,'Rotation',4X,'Moment, Mr',
*      3X,'Moment, Mt',3X,'Shear Force')
670 FORMAT(3X,' Ele Force, H1 Force, V1 Moment, M1 Force, H2
*Force, V2 Moment, M2')
680 FORMAT(3X,' Ele Force, H1 Force, V1 Force, H2 Force, V2')
690 FORMAT(/,5X,'Number of rotations taken in JACOBI =',I2,/)
700 FORMAT(/,5X,'EIGENVALUE(',I2,') = ',E14.6,2X,'SQRT(EGNVAL) = ',
*      E13.5,/,5X,'EIGENVECTOR:')
710 FORMAT(/,5X,'***** Number of time steps exceeded NTIME *****',/)
close(in)
close(it)
STOP
END

```

```

SUBROUTINE ASSEMBLE(NOD,MXELM,MXNEQ,NDF,NPE,NE,ITEM,GLK,GLM,GLF)

```

```

C -----
C
C   The subroutine is called in MAIN to assemble element coefficient
C   matrices (in a upper banded matrix form) and right-hand vectors
C
C   {ELF}.... Element source vector, {f}
C   {ELK}.... Element coefficient matrix, [K]
C   {ELM}.... Element coefficient matrix, [M]
C   [NOD].... Connectivity matrix, [B]
C -----

```

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```

C
  IMPLICIT REAL*8 (A-H,O-Z)
  DIMENSION  GLK(MXNEQ,MXNEQ),GLM(MXNEQ,MXNEQ),GLF(MXNEQ),
*            NOD(MXELM,4)
  COMMON/STF1/ELK(9,9),ELM(9,9),ELF(9),ELX(4),ELU(9),ELV(9),ELA(9)
  IF(ITEM.LE.2)THEN
C
C   Assemble element coefficient matrix ELK and source vector ELF
C
      DO 50 I = 1, NPE
      NR = (NOD(NE,I) - 1)*NDF
      DO 40 II = 1, NDF
      NR = NR + 1
      L = (I-1)*NDF + II
      GLF(NR) = GLF(NR) + ELF(L)
      DO 30 J = 1, NPE
      NCL = (NOD(NE,J)-1)*NDF
      DO 20 JJ = 1, NDF
      M = (J-1)*NDF + JJ
      NC = NCL-NR+JJ+1
      IF(NC)20,20,10
10    GLK(NR,NC) = GLK(NR,NC) + ELK(L,M)
20    CONTINUE
30    CONTINUE
40    CONTINUE
50    CONTINUE
      ELSE
C
C   ASSEMBLE ELEMENT MATRICES INTO FULL GLOBAL MATRICES
C
      DO 100 I=1,NPE
      NR=(NOD(NE,I)-1)*NDF
      DO 90 II=1,NDF
      NR=NR+1
      L=(I-1)*NDF+II
      DO 80 J=1,NPE
      NC=(NOD(NE,J)-1)*NDF
      DO 70 JJ=1,NDF
      M=(J-1)*NDF+JJ
      NC=NC+1
      GLK(NR,NC)=GLK(NR,NC)+ELK(L,M)
60    GLM(NR,NC)=GLM(NR,NC)+ELM(L,M)
70    CONTINUE
80    CONTINUE
90    CONTINUE
100   CONTINUE
C
      ENDIF
      RETURN
      END

```

```

SUBROUTINE BOUNDARY(NEQ,NEQR,NHBW,NSPV,NSSV,NNBC,NDF,DT,ITEM,ALFA,
*           IBDY,ISPV,ISSV,INBC,UREF,VSPV,VSSV,VNBC,
*           GLK,GLM,GLF,GUO,MXEBC,MXNBC,MXMBC,MXNEQ)
C -----
C
C The subroutine is called in MAIN to implement specified boundary
C conditions on the assembled system of finite element equations
C -----
C
C IMPLICIT REAL*8 (A-H,O-Z)
C DIMENSION ISPV(MXEBC,2),ISSV(MXNBC,2),INBC(MXMBC,2),IBDY(MXEBC)
C DIMENSION UREF(MXMBC),VSPV(MXEBC),VSSV(MXNBC),VNBC(MXMBC)
C DIMENSION GLK(MXNEQ,MXNEQ),GLM(MXNEQ,MXNEQ),GLF(MXNEQ),GUO(MXNEQ)
C
C Impose boundary conditions for STATIC and TIME-DEPENDENT problems
C
C IF(ITEM.LE.2)THEN
C
C Include specified PRIMARY degrees of freedom
C
C   IF(NSPV.NE.0)THEN
C     DO 30 NB = 1,NSPV
C       IE=(ISPV(NB,1)-1)*NDF+ISPV(NB,2)
C       IT=NHBW-1
C       I=IE-NHBW
C       DO 10 II=1,IT
C         I=I+1
C
C         IF(I .GE. 1)THEN
C           J=IE-I+1
C           GLF(I)=GLF(I)-GLK(I,J)*VSPV(NB)
C           GLK(I,J)=0.0
C         ENDIF
C   10 CONTINUE
C       GLK(IE,1)=1.0
C       GLF(IE)=VSPV(NB)
C       I=IE
C       DO 20 II=2,NHBW
C         I=I+1
C         IF(I .LE. NEQ)THEN
C           GLF(I)=GLF(I)-GLK(IE,II)*VSPV(NB)
C           GLK(IE,II)=0.0
C         ENDIF
C   20 CONTINUE
C   30 CONTINUE
C     ENDIF
C   IF(NSSV.NE.0)THEN
C

```

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```

C      Include specified SECONDARY degrees of freedom
C
      DO 40 NF = 1,NSSV
      NB=(ISSV(NF,1)-1)*NDF+ISSV(NF,2)
      IF (ITEM.EQ.1)GLF(NB)=GLF(NB)+VSSV(NF)*DT
40      IF (ITEM.NE.1)GLF(NB)=GLF(NB)+VSSV(NF)
      ENDIF
      IF (NNBC.NE.0) THEN
C
C      Include specified MIXED boundary conditions
C
      DO 50 IC=1,NNBC
      NC=(INBC(IC,1)-1)*NDF+INBC(IC,2)
      IF (ITEM.EQ.1) THEN
      GLK(NC,1)=GLK(NC,1)+ALFA*DT*VNBC(IC)
      GLF(NC)=GLF(NC)+DT*VNBC(IC)*(UREF(IC)
*          -(1.0-ALFA)*GUO(NC))
      ELSE
      GLK(NC,1)=GLK(NC,1)+VNBC(IC)
      GLF(NC)=GLF(NC)+VNBC(IC)*UREF(IC)
      ENDIF
50      CONTINUE
      ENDIF
      ELSE
C
C      Impose boundary conditions for EIGENVALUE problems
C
      IF (NNBC.NE.0) THEN
C
C      Include specified MIXED boundary conditions
C
      DO 70 IC=1,NNBC
      NC=(INBC(IC,1)-1)*NDF+INBC(IC,2)
      GLK(NC,NC)=GLK(NC,NC)+VNBC(IC)
70      CONTINUE
      ENDIF
C
C      Include specified PRIMARY degrees of freedom
C
      IF (NSPV.NE.0) THEN
      DO 80 IB=1,NSPV
80      IBDY(IB)=(ISPV(IB,1)-1)*NDF+ISPV(IB,2)
      DO 120 I=1,NSPV
      IMAX=IBDY(I)
      DO 110 J=I,NSPV
      IF (IBDY(J) .GE. IMAX) THEN
      IMAX=IBDY(J)
      IKEPT=J
      ENDIF
110      CONTINUE

```

```

        IBDY(IKEPT)=IBDY(I)
        IBDY(I)=IMAX
120     CONTINUE
        NEQR = NEQ
        DO 180 I=1,NSPV
        IB=IBDY(I)
        IF (IB .LT. NEQR) THEN
            NEQR1=NEQR-1
            DO 160 II=IB,NEQR1
            DO 140 JJ=1,NEQR
140         GLM(II,JJ)=GLM(II+1,JJ)
            GLK(II,JJ)=GLK(II+1,JJ)
            DO 150 JJ=1,NEQR
150         GLM(JJ,II)=GLM(JJ,II+1)
160         GLK(JJ,II)=GLK(JJ,II+1)
            CONTINUE
        ENDIF
        NEQR=NEQR-1
180     CONTINUE
        ENDIF
    ENDIF
    RETURN
    END

```

```

SUBROUTINE COEFFCNT(IELEM,ITEM,MODEL,NDF,NPE,TIME,NTYPE,
*              NE,F3,MXELM)

```

```

C -----
C
C The subroutine is called in MAIN to compute coefficient matrices
C and source vector for the model problem in Eq. (1) (see MAIN)
C
C X..... Global (i.e., problem) coordinate
C XI ..... Local (i.e., element) coordinate
C H..... Element length
C {SF}..... Element interpolation (or shape) functions
C {GDSF}.... First derivative of SF w.r.t. X
C {GDDSF}... Second derivative of SF w.r.t. X
C GJ..... Determinant of the Jacobian matrix
C [GAUSPT].. 4x4 matrix of Gauss points: N-th column corresponds
C to the N-point Gauss rule
C [GAUSWT].. 4x4 matrix of Gauss weights (see the comment above)
C [A],[B],.. Element matrices needed to compute ELK
C [ELK]..... Element coefficient matrix [K]
C [ELM]..... Element 'mass' matrix [M]
C -----

```

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```

C
  IMPLICIT REAL*8(A-H,O-Z)
  COMMON/STF1/ELK(9,9),ELM(9,9),ELF(9),ELX(4),ELU(9),ELV(9),ELA(9)
  COMMON/STF2/A1,A2,A3,A4,A5,AX0,AX1,BX0,BX1,CX0,CX1,CT0,CT1,FX0,
  *          FX1,FX2
  COMMON/SHP/SF(4),GDSF(4),GDDSF(4),GJ
  DIMENSION GAUSPT(5,5),GAUSWT(5,5),F3(MXELM)
C
  DATA GAUSPT/5*0.0D0,-.57735027D0,.57735027D0,3*0.0D0,-.77459667D0,
  * 0.0D0,.77459667D0,2*0.0D0,-.86113631D0,-.33998104D0,.33998104D0,
  *.86113631D0,0.0D0,-.906180D0,-.538469D0,0.0D0,.538469D0,.906180D0/
C
  DATA GAUSWT/2.0D0,4*0.0D0,2*1.0D0,3*0.0D0,.55555555D0,.88888888D0,
  * 0.55555555D0,2*0.0D0,.34785485D0,2*.65214515D0,.34785485D0,0.0D0,
  * 0.236927D0,.478629D0,.568889D0,.478629D0,.236927D0/
C
  NN=NDF*NPE
  H = ELX(NPE) - ELX(1)
  IF(IELEM.EQ.0)THEN
    NGP=4
  ELSE
    NGP = IELEM+1
  ENDIF
  DO 10 J=1,NN
    IF(ITEM.LE.2)THEN
      ELF(J) = 0.0
    ENDIF
    DO 10 I=1,NN
      IF(ITEM.GT.0)THEN
        ELM(I,J)=0.0
      ENDIF
    10 ELK(I,J)=0.0
C
  IF(MODEL.NE.2)THEN
C
  DO-LOOP on number of Gauss points begins here
C
    DO 100 NI=1,NGP
      XI = GAUSPT(NI,NGP)
C
  Call subroutine SHAPE1D to evaluate the interpolation functions
  and their global derivatives at the Gauss point XI
C
    CALL SHAPE1D(H,IELEM,NPE,XI)
    CONST = GJ*GAUSWT(NI,NGP)
    IF(IELEM.EQ.0)THEN
      X = ELX(1) + 0.5*H*(1.0+XI)
    ELSE
      X = 0.0
    DO 30 J=1,NPE

```


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```

        DO 60 J=1,NN
        IF (ITEM.LE.2) THEN
            ELF(J) = ELF(J) + CONST*SF(J)*FX*X
        ENDIF
        DO 60 I=1,NN
        IF (ITEM.NE.0) THEN
            ELM(I,J) = ELM(I,J) + CONST*SF(I)*SF(J)*CT*X
        ENDIF
        AIJ = CONST*GDSF(I)*GDSF(J)*C11*X
        CIJ = CONST*SF(I)*SF(J)*CX*X
        DIJ = CONST*(GDSF(I)*SF(J)+SF(I)*GDSF(J))*C12
        EIJ = CONST*SF(I)*SF(J)*C22/X
60      ELK(I,J)=ELK(I,J) + AIJ + CIJ + DIJ + EIJ
        ENDIF
    ELSE
C
C      Coefficients for the EULER-BERNOULLI theory (MODEL=2)
C
        IF (NTYPE.EQ.0) THEN
C
C      The Euler-Bernoulli BEAM element (MODEL=1 and NTYPE=0)
C
            BX=BX0+BX1*X
            CX=CX0+CX1*X
            DO 70 J = 1,NN
            IF (ITEM.LE.2) THEN
                ELF(J) = ELF(J) + CONST*SF(J)*FX
            ENDIF
            DO 70 I = 1,NN
            IF (ITEM.GT.0) THEN
                IF (ITEM.LE.3) THEN
                    ELM(I,J) = ELM(I,J) + CONST*SF(I)*SF(J)*CT
                ELSE
                    ELM(I,J) = ELM(I,J) + CONST*GDSF(I)*GDSF(J)
                ENDIF
            ENDIF
            BIJ = CONST*GDDSF(I)*GDDSF(J)
            CIJ = CONST*SF(I)*SF(J)
70      ELK(I,J)=ELK(I,J) + BX*BIJ + CX*CIJ
        ELSE
C
C      The E-B CIRCULAR PLATE element (MODEL=1 and NTYPE>0)
C
            ANU21=BX0*AX0/AX1
            DI=(BX1**3)/12.0
            D11=DI*AX0/(1.0-BX0*ANU21)
            D22=D11*(AX1/AX0)
            D12=BX0*D22
            DO 80 J=1,NN
            IF (ITEM.LE.2) THEN

```



```

      ELF(J) = ELF(J) + CONST*SF(J)*FX*X
    ENDIF
    DO 80 I=1,NN
      BIJ = CONST*GDDSF(I)*GDDSF(J)*D11*X
      CIJ = CONST*SF(I)*SF(J)*CX*X
      DIJ = CONST*(GDDSF(I)*GDSF(J)+GDSF(I)*GDDSF(J))*D12
      EIJ = CONST*GDSF(I)*GDSF(J)*D22/X
80    ELK(I,J)=ELK(I,J) + BIJ + CIJ + DIJ + EIJ
    ENDIF
  ENDIF
100 CONTINUE
  ELSE

C
C   Coefficients for the TIMOSHENKO beam and circular plate (MODEL=2)
C   Full integration for bending coefficients
C
    DO 160 NI=1,NGP
      XI=GAUSPT(NI,NGP)
      CALL SHAPE1D(H, IELEM,NPE,XI)
      CONST=GJ*GAUSWT(NI,NGP)
      X = 0.0
      DO 110 J=1,NPE
110    X = X + SF(J)*ELX(J)
        IF(NTYPE.EQ.0 .OR. NTYPE.EQ.2)THEN

C
C   The TIMOSHENKO BEAM element (MODEL=2 and NTYPE=0 OR 2)
C
          BX=BX0+BX1*X
          CX=CX0+CX1*X
          FX=FX0+FX1*X+FX2*X*X
          JJ=1
          DO 130 J=1,NPE
            IF(ITEM.LE.2)THEN
              ELF(JJ)=ELF(JJ)+FX*SF(J)*CONST
            ENDIF
            II=1
            DO 120 I=1,NPE
              CIJ=SF(I)*SF(J)*CONST
              BIJ=GDSF(I)*GDSF(J)*CONST
              ELK(II,JJ) =ELK(II,JJ) +CX*CIJ
              ELK(II+1,JJ+1)=ELK(II+1,JJ+1)+BX*BIJ
              IF(ITEM.NE.0)THEN
                ELM(II,JJ) =ELM(II,JJ) +CT0*CIJ
                ELM(II+1,JJ+1)=ELM(II+1,JJ+1)+CT1*CIJ
              ENDIF
120            II=NDF*I+1
130            JJ=NDF*J+1
            ELSE

```

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```

C
C   TIMOSHENKO CIRCULAR PLATE element (MODEL=2 and NTYPE=1 or 3)
C       AX0=E1, AX1=E2, BX0=ANU12, BX1=H
C
      ANU21=BX0*AX0/AX1
      CX=CX0+CX1*X
      FX=FX0+FX1*X
      DI=(BX1**3)/12.0
      D11=DI*AX0/(1.0-BX0*ANU21)
      D22=D11*(AX1/AX0)
      D12=BX0*D22
      JJ=1
      DO 150 J=1,NPE
      IF (ITEM.LE.2) THEN
          ELF(JJ)=ELF(JJ)+FX*SF(J)*CONST*X
      ENDIF
      II=1
      DO 140 I=1,NPE
      BIJ = CONST*GDSF(I)*GDSF(J)*D11*X
      CIJ = CONST*SF(I)*SF(J)*X
      DIJ = CONST*(GDSF(I)*SF(J)+SF(I)*GDSF(J))*D12
      EIJ = CONST*SF(I)*SF(J)*D22/X
      ELK(II,JJ) =ELK(II,JJ) + CX*CIJ
      ELK(II+1,JJ+1)=ELK(II+1,JJ+1) + BIJ + DIJ + EIJ
      IF (ITEM.NE.0) THEN
          ELM(II,JJ) =ELM(II,JJ) +CT0*CIJ
          ELM(II+1,JJ+1)=ELM(II+1,JJ+1)+CT1*CIJ
      ENDIF
140      II=NDF*I+1
150      JJ=NDF*J+1
      ENDIF
160      CONTINUE

C
C   Reduced integration is used to evaluate the transverse shear terms
C
      LGP=NGP-1
      DO 230 NI=1,LGP
      XI=GAUSPT(NI,LGP)
C
      CALL SHAPE1D(H,IELEM,NPE,XI)
      CONST=GJ*GAUSWT(NI,LGP)
C
      X = 0.0
      DO 170 J=1,NPE
170      X = X + SF(J)*ELX(J)
      IF (NTYPE.EQ.0 .OR. NTYPE.EQ.2) THEN
C
C       The TIMOSHENKO BEAM element (MODEL=2 and NTYPE=0 or 2)
C       AX = GAK = AX0 + AX1*X (reduced integration)

```

```

C
      AX=AX0+AX1*X
      JJ=1
      DO 190 J=1,NPE
      II=1
      DO 180 I=1,NPE
      B11=GDSF(I)*GDSF(J)*CONST
      B01=SF(I)*GDSF(J)*CONST
      B10=GDSF(I)*SF(J)*CONST
      B00=SF(I)*SF(J)*CONST
      ELK(II,JJ) =ELK(II,JJ) +AX*B11
      ELK(II,JJ+1) =ELK(II,JJ+1) +AX*B10
      ELK(II+1,JJ) =ELK(II+1,JJ) +AX*B01
      ELK(II+1,JJ+1)=ELK(II+1,JJ+1)+AX*B00
180      II=I*NDF+1
190      JJ=J*NDF+1
      ELSE
C
C      TIMOSHENKO CIRCULAR PLATE element (MODEL=2 and NTYPE=1 or 3)
C      BX1=H, FX2=G13*K (reduced integration)
C
      A33=BX1*FX2
      JJ=1
      DO 210 J=1,NPE
      II=1
      DO 200 I=1,NPE
      BIJ = CONST*GDSF(I)*GDSF(J)*X
      CIJ = CONST*SF(I)*SF(J)*X
      DIJ = CONST*GDSF(I)*SF(J)*X
      DJI = CONST*SF(I)*GDSF(J)*X
      ELK(II,JJ) =ELK(II,JJ) + A33*BIJ
      ELK(II,JJ+1) =ELK(II,JJ+1) + A33*DIJ
      ELK(II+1,JJ) =ELK(II+1,JJ) + A33*DJI
      ELK(II+1,JJ+1)=ELK(II+1,JJ+1) + A33*CIJ
200      II=NDF*I+1
210      JJ=NDF*J+1
      ENDIF
230      CONTINUE
      IF (ITEM.EQ.0 .AND. NTYPE.GT.1) THEN
      CALL TIMFORCE(ELF,ELX,FX0,FX1,FX2,H,NTYPE,NE,F3,MXELM)
      ENDIF
      ENDIF
C
      IF (ITEM.GT.2) RETURN
      IF (ITEM.EQ.1 .OR. ITEM.EQ.2) THEN
C
C      Equivalent coefficient matrices for TIME-DEPENDENT problems
C
      IF (ITEM .EQ. 1) THEN
      IF (ITEM .EQ. 1) THEN

```

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```

C
C   Alfa-family of time approximation for PARABOLIC equations
C
      DO 250 J=1,NN
      SUM=0.0
      DO 240 I=1,NN
      SUM=SUM+(ELM(I,J)-A2*ELK(I,J))*ELU(I)
240     ELK(I,J)=ELM(I,J)+A1*ELK(I,J)
250     ELF(J)=(A1+A2)*ELF(J)+SUM
      ELSE
C
C   Newmark-family of time approximation for HYPERBOLIC equations
C
      IF (TIME.EQ.0.0) THEN
      DO 260 J=1,NN
      DO 260 I=1,NN
      ELF(J)=ELF(J)-ELK(I,J)*ELU(I)
260     ELK(I,J)=ELM(I,J)
      ELSE
      DO 270 J=1,NN
      DO 270 I=1,NN
      ELF(J)=ELF(J)+ELM(I,J)*(A3*ELU(I)+A4*ELV(I)+A5*ELA(I))
270     ELK(I,J)=ELK(I,J)+A3*ELM(I,J)
      ENDIF
      ENDIF
      ENDIF
      RETURN
      END

      SUBROUTINE CONSTRNT (NEQ, NHBW, NDF, NCON, ICON, VCON, GLK, GLM, GLF,
*          TRM, MXNEQ)
C
C   -----
C
C   The subroutine is called in MAIN to implement specified constraint
C   conditions (e.g., inclined supports) on the condensed system of
C   equations. Array GLM is used here as a temporary storage array.
C   -----
C
      IMPLICIT REAL*8 (A-H,O-Z)
      DIMENSION  ICON(9), VCON(9), GLK(MXNEQ, MXNEQ), GLF(MXNEQ),
*          GLM(MXNEQ, MXNEQ), TRM(MXNEQ, MXNEQ)
      PI=3.14159265D0
C
C   Include specified constraint conditions
C
      DO 20 IC=1, NEQ
      DO 10 JC=1, NEQ
      GLM(IC, JC)=0.0
10     TRM(IC, JC)=0.0

```

```

20   TRM(IC, IC)=1.0D0
DO 30 IC=1, NCON
   BETA=VCON(IC)*PI/180.0D0
   IDOF=NDF*ICON(IC)-1
   TRM(IDOF, IDOF) = DCOS(BETA)
   TRM(IDOF, IDOF+1) = DSIN(BETA)
   TRM(IDOF+1, IDOF) = -DSIN(BETA)
30   TRM(IDOF+1, IDOF+1)= DCOS(BETA)
   L=0
   DO 50 I=1, NEQ
   DO 40 J=1, NHBW
40   GLM(I, L+J)=GLK(I, J)
50   L=L+1
   DO 60 I=1, NEQ
   DO 60 J=I, NEQ
60   GLM(J, I)=GLM(I, J)
   DO 70 I=1, NEQ
   DO 70 J=1, NEQ
70   GLK(I, J)=GLM(I, J)
   DO 80 I=1, NEQ
   DO 80 J=1, NEQ
   GLM(I, J)=0.0
   DO 80 K=1, NEQ
80   GLM(I, J)=GLM(I, J)+TRM(I, K)*GLK(K, J)
   DO 90 I=1, NEQ
   DO 90 J=1, NEQ
   GLK(I, J)=0.0
   DO 90 K=1, NEQ
90   GLK(I, J)=GLK(I, J)+GLM(I, K)*TRM(J, K)
   DO 100 I=1, NEQ
   DO 100 J=1, NEQ
100  TRM(I, J)=GLK(I, J)
   L=0
   DO 120 I=1, NEQ
   DO 110 J=1, NHBW
110  GLK(I, J)=TRM(I, L+J)
120  L=L+1
   DO 150 I=1, NEQ
   GLM(I, 1)=0.0
   DO 140 K=1, NEQ
140  GLM(I, 1)=GLM(I, 1)+TRM(I, K)*GLF(K)
150  GLF(I)=GLM(I, 1)
   RETURN
   END

```

```

SUBROUTINE ECHODATA(IN, IT)
IMPLICIT REAL*8(A-H, O-Z)
DIMENSION AA(20)
WRITE(IT, 40)

```

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```

10 CONTINUE
   READ(IN,30,END=20) AA
   WRITE(IT,60) AA
   GO TO 10
20 CONTINUE
   REWIND(IN)
   WRITE(IT,50)
   RETURN
30 FORMAT(20A4)
40 FORMAT(5X,'*** ECHO OF THE INPUT DATA STARTS ***',/)
50 FORMAT(5X,'**** ECHO OF THE INPUT DATA ENDS ****',/)
60 FORMAT(1X,20A4)
   END

      SUBROUTINE EGN SOLVR(N,A,B,XX,X,NEGN,NR,MXNEQ)
C -----
C
C   The subroutine is called in MAIN to solve the EIGENVALUE PROBLEM
C
C           [A]{X} = Lambda[B]{X}
C
C   The program can be used only for positive-definite [B] matrix.
C   The dimensions of V, VT, W, and IH should be equal to MXNEQ.
C -----
C
      IMPLICIT REAL*8 (A-H,O-Z)
      DIMENSION A(MXNEQ,MXNEQ),B(MXNEQ,MXNEQ),XX(MXNEQ),X(MXNEQ,MXNEQ)
      DIMENSION V(500,500),VT(500,500),W(500,500),IH(500)
C
      Call subroutine JACOBI to diagonalize [B]
C
      CALL JACOBI (N,B,NEGN,NR,V,XX,IH,MXNEQ)
C
      Make diagonalized [B] symmetric
C
      DO 10 I=1,N
      DO 10 J=1,N
10 B(J,I)=B(I,J)
C
      Check (to make sure) that [B] is positive-definite
C
      DO 30 I=1,N
      IF (B(I,I))20,30,30
20 WRITE(6,80)
      STOP
30 CONTINUE
C   The eigenvectors of [B] are stored in array V(I,J)
C   Form the transpose of [V] as [VT]
C

```

```

      DO 40 I=1,N
      DO 40 J=1,N
40  VT(I,J)=V(J,I)
C
C   Find the product [F]=[VT][A][V] and store in [A] to save storage
C
      CALL MATRXMLT (MXNEQ,N,VT,A,W)
      CALL MATRXMLT (MXNEQ,N,W,V,A)
C
C   Get [GI] from diagonalized [B], but store it in [B]
C
      DO 50 I=1,N
50  B(I,I)=1.0/DSQRT(B(I,I))
C
C   Find the product [Q]=[GI][F][GI]=[B][A][B] and store in [A]
C
      CALL MATRXMLT (MXNEQ,N,B,A,W)
      CALL MATRXMLT (MXNEQ,N,W,B,A)
C
C   We now have the form [Q]{Z}=Lamda{Z}. Diagonalize [Q] to obtain
C   the eigenvalues by calling JACOBI. The eigenvalues are returned
C   as diag [A].
C
      CALL JACOBI (N,A,NEGN,NR,VT,XX,IH,MXNEQ)
      DO 60 J=1,N
60  XX(J)=A(J,J)
C
C   The eigenvectors are computed from the relation,
C           {X}=[V][GI]{Z}=[V][B][VT]
C   since {Z} is stored in [VT]
C
      CALL MATRXMLT (MXNEQ,N,V,B,W)
      CALL MATRXMLT (MXNEQ,N,W,VT,X)
C
80  FORMAT(/'*** Matrix [GLM] is NOT positive-definite ***')
      RETURN
      END

      SUBROUTINE EQNSOLVR(NRM,NCM,NEQNS,NBW,BAND,RHS,IRES)
C
C   -----
C
C   The subroutine is called in MAIN to solve symmetric and banded set
C   of equations using the Gauss elimination method:[BAND]{U} = {RHS}.
C   The coefficient matrix is input as BAND(NEQNS,NBW) and the column
C   vector is input as RHS(NEQNS), where NEQNS is the actual number
C   of equations and NBW is the half band width. The true dimensions
C   of the matrix [BAND] in the calling program, are NRM by NCM. When
C   IRES is greater than zero, the right hand elimination is skipped.
C
C   -----

```

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```

C
  IMPLICIT REAL*8(A-H,O-Z)
  DIMENSION BAND(NRM,NCM),RHS(NRM)
C
  MEQNS=NEQNS-1
  IF(IRES.LE.0) THEN
    DO 30 NPIV=1,MEQNS
      NPIVOT=NPIV+1
      LSTSUB=NPIV+NBW-1
      IF(LSTSUB.GT.NEQNS) THEN
        LSTSUB=NEQNS
      ENDIF
C
      DO 20 NROW=NPIVOT,LSTSUB
        NCOL=NROW-NPIV+1
        FACTOR=BAND(NPIV,NCOL)/BAND(NPIV,1)
        DO 10 NCOL=NROW,LSTSUB
          ICOL=NCOL-NROW+1
          JCOL=NCOL-NPIV+1
10      BAND(NROW,ICOL)=BAND(NROW,ICOL)-FACTOR*BAND(NPIV,JCOL)
20      RHS(NROW)=RHS(NROW)-FACTOR*RHS(NPIV)
30      CONTINUE

    ELSE
40      DO 60 NPIV=1,MEQNS
        NPIVOT=NPIV+1
        LSTSUB=NPIV+NBW-1
        IF(LSTSUB.GT.NEQNS) THEN
          LSTSUB=NEQNS
        ENDIF
        DO 50 NROW=NPIVOT,LSTSUB
          NCOL=NROW-NPIV+1
          FACTOR=BAND(NPIV,NCOL)/BAND(NPIV,1)
50      RHS(NROW)=RHS(NROW)-FACTOR*RHS(NPIV)
60      CONTINUE
    ENDIF
C
C    Back substitution
C
  DO 90 IJK=2,NEQNS
    NPIV=NEQNS-IJK+2
    RHS(NPIV)=RHS(NPIV)/BAND(NPIV,1)
    LSTSUB=NPIV-NBW+1
    IF(LSTSUB.LT.1) THEN
      LSTSUB=1
    ENDIF
    NPIVOT=NPIV-1

```



```

DO 80 JKI=LSTSUB,NPIVOT
NROW=NPIVOT-JKI+LSTSUB
NCOL=NPIV-NROW+1
FACTOR=BAND(NROW,NCOL)
80 RHS(NROW)=RHS(NROW)-FACTOR*RHS(NPIV)
90 CONTINUE
RHS(1)=RHS(1)/BAND(1,1)
RETURN
END

```

```

SUBROUTINE JACOBI (N,Q,JVEC,M,V,X,IH,MXNEQ)

```

```

C -----
C
C   Called in EGNLSOLVR to diagonalize [Q] by successive rotations
C
C   DESCRIPTION OF THE VARIABLES:
C
C   N   .... Order of the real, symmetric matrix [Q] (N > 2)
C   [Q] .... The matrix to be diagonalized (destroyed)
C   JVEC .... 0, when only eigenvalues alone have to be found
C   [V] .... Matrix of eigenvectors
C   M   .... Number of rotations performed
C -----
C
IMPLICIT REAL*8 (A-H,O-Z)
DIMENSION Q(MXNEQ,MXNEQ),V(MXNEQ,MXNEQ),X(MXNEQ),IH(MXNEQ)
EPSI=1.0D-08
IF(JVEC)10,50,10
10 DO 40 I=1,N
DO 40 J=1,N
IF(I-J)30,20,30
20 V(I,J)=1.0
GO TO 40
30 V(I,J)=0.0
40 CONTINUE
50 M=0
MI=N-1
DO 70 I=1,MI
X(I)=0.0
MJ=I+1
DO 70 J=MJ,N
IF(X(I)-DABS(Q(I,J)))60,60,70
60 X(I)=DABS(Q(I,J))
IH(I)=J
70 CONTINUE
75 DO 100 I=1,MI
IF(I-1)90,90,80
80 IF(XMAX-X(I))90,100,100
90 XMAX=X(I)

```

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```

      IP=I
      JP=IH(I)
100 CONTINUE
      IF(XMAX-EPST)500,500,110
110 M=M+1
      IF(Q(IP,IP)-Q(JP,JP))120,130,130
120 TANG=-2.0*Q(IP,JP)/(DABS(Q(IP,IP)-Q(JP,JP))+DSQRT((Q(IP,IP)
1      -Q(JP,JP))**2+4.0*Q(IP,JP)**2))
      GO TO 140
130 TANG= 2.0*Q(IP,JP)/(DABS(Q(IP,IP)-Q(JP,JP))+DSQRT((Q(IP,IP)
1      -Q(JP,JP))**2+4.0*Q(IP,JP)**2))
140 COSN=1.0/DSQRT(1.0+TANG**2)
      SINE=TANG*COSN
      QII=Q(IP,IP)
      Q(IP,IP)=COSN**2*(QII+TANG*(2.*Q(IP,JP)+TANG*Q(JP,JP)))
      Q(JP,JP)=COSN**2*(Q(JP,JP)-TANG*(2.*Q(IP,JP)-TANG*QII))
      Q(IP,JP)=0.0
      IF (Q(IP,IP)-Q(JP,JP)) 150,190,190
150 TEMP=Q(IP,IP)
      Q(IP,IP)=Q(JP,JP)
      Q(JP,JP)=TEMP
      IF(SINE) 160,170,170
160 TEMP=COSN
      GOTO 180
170 TEMP=-COSN
180 COSN=DABS(SINE)
      SINE=TEMP
190 DO 260 I=1,MI
      IF (I-IP) 210,260,200
200 IF (I-JP) 210,260,210
210 IF (IH(I)-IP) 220,230,220
220 IF (IH(I)-JP) 260,230,260
230 K=IH(I)
      TEMP=Q(I,K)
      Q(I,K)=0.0
      MJ=I+1
      X(I)=0.0
      DO 250 J=MJ,N
      IF (X(I)-DABS(Q(I,J))) 240,240,250
240 X(I)=DABS(Q(I,J))
      IH(I)=J
250 CONTINUE
      Q(I,K)=TEMP
260 CONTINUE
      X(IP)=0.0
      X(JP)=0.0
      DO 430 I=1,N
      IF(I-IP) 270,430,320
270 TEMP=Q(I,IP)
      Q(I,IP)=COSN*TEMP+SINE*Q(I,JP)

```

```

      IF (X(I)-DABS(Q(I,IP))) 280,290,290
280 X(I)=DABS(Q(I,IP))
      IH(I)=IP
290 Q(I,JP)=-SINE*TEMP+COSN*Q(I,JP)
      IF (X(I)-DABS(Q(I,JP))) 300,430,430
300 X(I)=DABS(Q(I,JP))
      IH(I)=JP
      GO TO 430
320 IF(I-JP) 330,430,380
330 TEMP=Q(IP,I)
      Q(IP,I)=COSN*TEMP+SINE*Q(I,JP)
      IF(X(IP)-DABS(Q(IP,I)))340,350,350
340 X(IP)=DABS(Q(IP,I))
      IH(IP)=I
350 Q(I,JP)=-SINE*TEMP+COSN*Q(I,JP)
      IF (X(I)-DABS(Q(I,JP))) 300,430,430
380 TEMP=Q(IP,I)
      Q(IP,I)=COSN*TEMP+SINE*Q(JP,I)
      IF(X(IP)-DABS(Q(IP,I)))390,400,400
390 X(IP)=DABS(Q(IP,I))

      IH(IP)=I
400 Q(JP,I)=-SINE*TEMP+COSN*Q(JP,I)
      IF(X(JP)-DABS(Q(JP,I)))410,430,430
410 X(JP)=DABS(Q(JP,I))
      IH(JP)=I
430 CONTINUE
      IF(JVEC)440,75,440
440 DO 450 I=1,N
      TEMP=V(I,IP)
      V(I,IP)=COSN*TEMP+SINE*V(I,JP)
450 V(I,JP)=-SINE*TEMP+COSN*V(I,JP)
      GOTO 75
500 RETURN
      END

```

```

SUBROUTINE MATRXMLT(MXNEQ,N,A,B,C)

```

```

C -----
C
C Called in EGNLSOLVR to compute the product of matrices [A] & [B]:
C [C]=[A][B]
C -----
C
C IMPLICIT REAL*8 (A-H,O-Z)
C DIMENSION A(MXNEQ,MXNEQ),B(MXNEQ,MXNEQ),C(MXNEQ,MXNEQ)
C DO 10 I=1,N
C DO 10 J=1,N
C C(I,J)=0.0

```

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```

      DO 10 K=1,N
10  C(I,J)=C(I,J)+A(I,K)*B(K,J)
      RETURN
      END

      SUBROUTINE MESH1D(NEM,NPE,NOD,MXELM,MXNOD,DX,GLX)
C
C -----
C   The subroutine is called in MAIN to compute arrays {GLX} and [NOD]
C
C   {GLX}.... Vector of global coordinates
C   {DX}..... Vector of element lengths [DX(1) = node 1 coordinate]
C   [NOD].... Connectivity matrix
C -----
C
      IMPLICIT REAL*8 (A-H,O-Z)
      DIMENSION GLX(MXNOD),DX(MXNOD),NOD(MXELM,4)
C
C   Generate the elements of the connectivity matrix
C
      DO 10 I=1,NPE
10  NOD(1,I)=I
      DO 20 N=2,NEM
      DO 20 I=1,NPE
20  NOD(N,I) = NOD(N-1,I)+NPE-1
C
C   Generate global coordinates of the global nodes
C
      GLX(1)=DX(1)
      IF(NPE.EQ.2)THEN
        DO 30 I=1,NEM
30   GLX(I+1) = GLX(I) + DX(I+1)
      ELSE
        DO 40 I=1,NEM
          II=2*I
          GLX(II) = GLX(II-1) + 0.5*DX(I+1)
40   GLX(II+1)=GLX(II-1) + DX(I+1)
      ENDIF
      RETURN
      END

      SUBROUTINE POSTPROC(DCAX,DCBX,DCCX,F3,GLF,GLX,NOD,ICONT,IELEM,NPE,
*      MODEL,NTYPE,ITEM,MXELM,MXNEQ,MXNOD,NEM,NDF)
C
C -----
C   The subroutine is called in MAIN to compute the solution and its
C   derivatives at five points, including the nodes of the element.
C   The bending moment (BM) and shear force (VF) are computed as per
C   the definitions given in Fig. 5.2.1 and Eq. (5.3.1) of the book;

```

```

C
C      X..... Global (i.e., problem) coordinate
C      XI ..... Local (i.e., element) coordinate
C      SF..... Element interpolation (or shape) functions
C      GDSF.... First derivative of SF w.r.t. global coordinate
C      GDDSF... Second derivative of SF w.r.t. global coordinate
C      ELU..... Element solution vector
C      U..... Interpolated solution
C      DU..... Interpolated derivative of the solution
C      W..... Interpolated transverse deflection
C      S..... Interpolated rotation function
C      DS..... Interpolated derivative of the rotation
C      DW..... Interpolated derivative of the transverse deflection
C      DDW..... Interpolated second derivative of trans. deflection
C      DDDW..... Interpolated third derivative of trans. deflection
C
C      -----
C
C      IMPLICIT REAL*8 (A-H,O-Z)
C      DIMENSION DCAX(MXELM,2),DCBX(MXELM,2),DCCX(MXELM,2)
C      DIMENSION F3(MXELM),GLF(MXNEQ),GLX(MXNOD),NOD(MXELM,4)
C      DIMENSION XP(9),ELX(4),ELU(9)
C      COMMON/IO/IN,IT
C      COMMON/SHP/SF(4),GDSF(4),GDDSF(4),GJ
C      COMMON/STF2/A1,A2,A3,A4,A5,AX0,AX1,BX0,BX1,CX0,CX1,CT0,CT1,FX0,
C      *          FX1,FX2
C      DATA XP/-1.0D0, -0.750D0, -0.50D0, -0.250D0, 0.0D0, 0.250D0,
C      *          0.50D0, 0.750D0, 1.0D0/
C
C      NPTS=9
C      DO 80 NE = 1, NEM
C      IF(ICONT.NE.1) THEN
C          AX0=DCAX(NE,1)
C          AX1=DCAX(NE,2)
C          BX0=DCBX(NE,1)
C          BX1=DCBX(NE,2)
C          CX0=DCCX(NE,1)
C          CX1=DCCX(NE,2)
C      ENDIF
C      L=0
C      DO 10 I=1,NPE
C      NI=NOD(NE,I)
C      IF(ICONT.NE.1) THEN
C          ELX(1)=0.0
C          ELX(2)=0.5*GLX(NE)
C          ELX(NPE)=GLX(NE)
C      ELSE
C          ELX(I)=GLX(NI)
C      ENDIF
C      LI=(NI-1)*NDF
C      DO 10 J=1,NDF

```

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```

      LI=LI+1
      L=L+1
10  ELU(L)=GLF(LI)
      H = ELX(NPE) - ELX(1)
C
      DO 70 NI=1,NPTS
      XI = XP(NI)
      CALL SHAPE1D(H, IELEM,NPE, XI)

      IF(MODEL.EQ.3)THEN
      W=0.0
      DW=0.0
      DDW=0.0
      XC=ELX(1)+0.5*H*(1.0+XI)
      DO 20 I=1,4
      W =W + SF(I)*ELU(I)
      DW =DW + GDSF(I)*ELU(I)
20  DDW=DDW+ GDSF(I)*ELU(I)
      DDDW=((ELU(1)-ELU(3))*2.0/H-(ELU(4)+ELU(2)))*6.0/(H*H)
      THETA=-DW
      IF(NTYPE.EQ.0)THEN
      BM=-(BX0+XC*BX1)*DDW
      VF=-(BX0+XC*BX1)*DDDW - BX1*DDW
      WRITE(IT,90)XC,W,THETA,BM,VF
      ELSE
      ANU21=BX0*AX0/AX1
      DI=(BX1**3)/12.0
      D11=DI*AX0/(1.0-BX0*ANU21)
      D22=D11*(AX1/AX0)
      D12=BX0*D22
      BMR=-(D11*DDW*XC+D12*DW)
      BMT=-(D12*DDW*XC+D22*DW)
      IF(XC.NE.0.0)THEN
      SFV=-D11*(XC*DDDW+DDW)+D22*DW/XC
      WRITE(IT,90)XC,W,THETA,BMR,BMT,SFV
      ELSE
      WRITE(IT,90)XC,W,THETA,BMR,BMT
      ENDIF
      ENDIF
      ELSE
      XC=0.0
      DO 30 I=1,NPE
30  XC=XC+SF(I)*ELX(I)
      IF(MODEL.EQ.1)THEN
      U=0.0
      DU=0.0
      DO 40 I=1,NPE
      U=U+SF(I)*ELU(I)
40  DU=DU+GDSF(I)*ELU(I)

```

```

IF(NTYPE.EQ.0)THEN
  SV=(AX0+AX1*XC)*DU
  WRITE(IT,90)XC,U,SV
ELSE
  ANU21=BX0*AX0/AX1
  IF(NTYPE.EQ.1)THEN
    C11=BX1*AX0/(1.0-BX0*ANU21)
    C22=C11*(AX1/AX0)
    C12=BX0*C22
  ELSE
    DENOM=1.0-BX0-ANU21
    C11=BX1*AX0*(1.0-BX0)/(1.0+BX0)/DENOM
    C22=BX1*AX1*(1.0-ANU21)/(1.0+ANU21)/DENOM
    C12=BX0*C22
  ENDIF
  IF(XC.NE.0.0)THEN
    SR=C11*DU+C12*U/XC
    ST=C12*DU+C22*U/XC
    WRITE(IT,90)XC,U,SR,ST
  ELSE
    WRITE(IT,90)XC,U,DU
  ENDIF
ENDIF
ELSE
C
C MODEL.EQ.2 Calculations
C
  IF(ITEM.EQ.0 .AND. NTYPE.GT.1)THEN
    H=ELX(NPE)-ELX(1)
    CALL TIMSTRES(AX0,ELU,XI,W,DW,SI,DSI,NE,F3,H,MXELM)
  ELSE
    W =0.0
    DW =0.0
    PSI =0.0
    DPSI =0.0
    DO 50 I=1,NPE
      L=2*I-1
      W = W +SF(I)*ELU(L)
      DW = DW +GDSF(I)*ELU(L)
      PSI = PSI +SF(I)*ELU(L+1)
      DPSI= DPSI+GDSF(I)*ELU(L+1)
    50
  ENDIF
  IF(NTYPE.EQ.0 .OR. NTYPE.EQ.2)THEN
    BM=(BX0+BX1*XC)*DPSI
    VF=(AX0+AX1*XC)*(DW+PSI)
    WRITE(IT,90)XC,W,PSI,BM,VF
  ELSE
    ANU21=BX0*AX0/AX1
    DI =(BX1**3)/12.0
    D11=DI*AX0/(1.0-BX0*ANU21)

```

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```

          D22=D11*(AX1/AX0)
          D12=BX0*D22
          BMR=(D11*DPSI*XC+D12*PSI)
          BMT=(D12*DPSI*XC+D22*PSI)
          SFV=FX2*(DW+PSI)*XC
          WRITE(IT,90)XC,W,PSI,BMR,BMT,SFV
        ENDIF
      ENDIF
    ENDIF
70 CONTINUE
80 CONTINUE
    RETURN
90 FORMAT(2X,6E13.5)
    END

SUBROUTINE REACTION(MXELEM,MXNEQ,NDF,NEM,NOD,NPE,NTYPE,PR,GLF,
*                SE,SL,SA,SI,CS,SN,CNT,SNT,HF,VF,PF,XB)
C
C -----
C   The subroutine is called in MAIN to compute generalized reaction
C   forces in each element of truss (NDF=2) or frame (NDF=3) structure
C -----
C
    IMPLICIT REAL*8(A-H,O-Z)
    DIMENSION PR(MXELEM),SE(MXELEM),SL(MXELEM),SA(MXELEM),SI(MXELEM)
    DIMENSION CS(MXELEM),SN(MXELEM),CNT(MXELEM),SNT(MXELEM)
    DIMENSION HF(MXELEM),VF(MXELEM),PF(MXELEM),XB(MXELEM)
    DIMENSION NOD(MXELEM,4),GLF(MXNEQ),ELR(6)
    COMMON/STF1/ELK(9,9),ELM(9,9),ELF(9),ELX(4),ELU(9),ELV(9),ELA(9)
C
    NN=NPE*NDF
    DO 140 N=1,NEM
        CN1=CS(N)
        SN1=SN(N)
C
C   Call TRANSFRM to compute element stiffness matrix and force vector
C
        L=0
        DO 100 I=1,NPE
            NI=NOD(N,I)
            LI=(NI-1)*NDF
            DO 100 J=1,NDF
                LI=LI+1
            L=L+1
100 ELU(L)=GLF(LI)
            CALL TRANSFRM(MXELEM,N,NTYPE,PR,SE,SL,SA,SI,CS,SN,
*                CNT,SNT,HF,VF,PF,XB)
C
C   Compute the FORCE and MOMENT RESULTANTS

```



```

C
DO 120 I=1,NN
  ELR(I) = 0.0
DO 110 J=1,NN
110 ELR(I) = ELR(I) + ELK(I,J)*ELU(J)
120 ELR(I) = ELR(I) - ELF(I)
  ELF(1) = ELR(1)*CN1+ELR(2)*SN1
  ELF(2) = -ELR(1)*SN1+ELR(2)*CN1
  IF(NTYPE.NE.0) THEN
    ELF(3) = ELR(3)
    ELF(4) = ELR(4)*CN1+ELR(5)*SN1
    ELF(5) = -ELR(4)*SN1+ELR(5)*CN1
    ELF(6) = ELR(6)
  ELSE
    ELF(3) = ELR(3)*CN1+ELR(4)*SN1
    ELF(4) = -ELR(3)*SN1+ELR(4)*CN1
  ENDIF
  WRITE(6,150)N, (ELF(I),I=1,NN)
  WRITE(6,160) (ELR(I),I=1,NN)
140 CONTINUE
  RETURN
150 FORMAT (3X,I2,6E12.4)
160 FORMAT (5X,6E12.4,/)
END

SUBROUTINE SHAPE1D(H,IELEM,NPE,XI)
C
C -----
C
C   Called in MAIN to compute shape functions and their derivatives
C   for Hermite cubic and Lagrange linear, quadratic and cubic elements
C
C   X..... Global (i.e., problem) coordinate
C   XI ..... Local (i.e., element) coordinate
C   H..... Element length
C   {SF}..... Interpolation (or shape) functions
C   {DSF}..... First derivative of SF w.r.t. XI
C   {DDSF}.... Second derivative of SFH w.r.t. XI
C   {GDSF}.... First derivative of SF w.r.t. X
C   {GDDSF}... Second derivative of SFH w.r.t. X
C   GJ..... Determinant of the Jacobian matrix
C
C -----
C
  IMPLICIT REAL*8 (A-H,O-Z)
  COMMON/SHP/SF(4),GDSF(4),GDDSF(4),GJ
  DIMENSION DSF(4),DDSF(4)
  IF(IELEM.EQ.0)THEN
C
C   HERMITE interpolation functions (for the Euler-Bernoulli theory)

```

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```

C
      NET=4
      SF(1) = 0.25*(2.0-3.0*XI+XI**3)
      SF(2) = -H*(1.0-XI)*(1.0-XI*XI)/8.0
      SF(3) = 0.25*(2.0+3.0*XI-XI**3)
      SF(4) = H*(1.0+XI)*(1.0-XI*XI)/8.0
      DSF(1) = -0.75*(1.0-XI*XI)
      DSF(2) = H*(1.0+2.0*XI-3.0*XI*XI)/8.0
      DSF(3) = 0.75*(1.0-XI*XI)
      DSF(4) = H*(1.0-2.0*XI-3.0*XI*XI)/8.0
      DDSF(1)= 1.5*XI
      DDSF(2)= 0.25*H*(1.0-3.0*XI)
      DDSF(3)= -1.5*XI
      DDSF(4)= -0.25*(1.0+3.0*XI)*H
    ELSE
      NET=NPE
      IF(IELEM.EQ.1)THEN

C
C   LAGRANGE interpolation functions used for linear, quadratic and
C   cubic approximation of second-order equations
C
C   LINEAR interpolation functions
C
      SF(1) = 0.5*(1.0-XI)
      SF(2) = 0.5*(1.0+XI)
      DSF(1) = -0.5
      DSF(2) = 0.5
      DDSF(1)= 0.0
      DDSF(2)= 0.0
    ELSE
      IF(IELEM.EQ.2)THEN

C
C   QUADRATIC interpolation functions
C
      SF(1) = -0.5*XI*(1.0-XI)
      SF(2) = 1.0-XI*XI
      SF(3) = 0.5*XI*(1.0+XI)
      DSF(1) = -0.5*(1.0-2.0*XI)
      DSF(2) = -2.0*XI
      DSF(3) = 0.5*(1.0+2.0*XI)
      DDSF(1)= 1.0
      DDSF(2)= -2.0
      DDSF(3)= 1.0
    ELSE

C
C   CUBIC interpolation functions
C
      SF(1) = 0.0625*(1.0-XI)*(9.0*XI*XI-1.)
      SF(2) = 0.5625*(1.0-XI*XI)*(1.0-3.0*XI)

```

```

SF(3) = 0.5625*(1.0-XI*XI)*(1.0+3.0*XI)
SF(4) = 0.0625*(9.0*XI*XI-1.0)*(1.0+XI)
DSF(1) = 0.0625*(1.0+18.0*XI-27.0*XI*XI)
DSF(2) = 0.5625*(-3.0-2.0*XI+9.0*XI*XI)
DSF(3) = 0.5625*(3.0-2.0*XI-9.0*XI*XI)
DSF(4) = 0.0625*(18.0*XI+27.0*XI*XI-1.0)
DDSF(1) = 0.0625*(18.0-54.0*XI)
DDSF(2) = 0.5625*(-2.0+18.0*XI)
DDSF(3) = 0.5625*(-2.0-18.0*XI)
DDSF(4) = 0.0625*(18.0+54.0*XI)
ENDIF
ENDIF
ENDIF
C
C   Compute derivatives of the interpolation functions w.r.t. X
C
80 GJ = H*0.5
DO 90 I = 1,NET
  GDSF(I) = DSF(I)/GJ
90 GDDSF(I) = DDSF(I)/GJ/GJ
RETURN
END

SUBROUTINE TIMFORCE(ELF,ELX,FX0,FX1,FX2,H,NTYPE,NE,F3,MXELM)
C
C -----
C   Called in COEFFCNT to compute element force vector for the
C   consistent interpolation Timoshenko element (CIE)
C -----
C
IMPLICIT REAL*8(A-H,O-Z)
COMMON/SHP/SF(4),GDSF(4),GDDSF(4),GJ
DIMENSION GAUSPT(5,5),GAUSWT(5,5),ELF(9),ELX(4),EX(3),F3(MXELM)
C
DATA GAUSPT/5*0.0D0,-.57735027D0,.57735027D0,3*0.0D0,-.77459667D0,
* 0.0D0,.77459667D0,2*0.0D0,-.86113631D0,-.33998104D0,.33998104D0,
*.86113631D0,0.0D0,-.906180D0,-.538469D0,0.0D0,.538469D0,.906180D0/
C
DATA GAUSWT/2.0D0,4*0.0D0,2*1.0D0,3*0.0D0,.55555555D0,.88888888D0,
* 0.55555555D0,2*0.0D0,.34785485D0,2*.65214515D0,.34785485D0,0.0D0,
* 0.236927D0,.478629D0,.568889D0,.478629D0,.236927D0/

NPE=3
IEL=2
NDF=2
NGP=IEL+1
DO 10 I=1,6
10 ELF(I)=0.0

```

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```

C
  EX(1)=ELX(1)
  EX(2)=ELX(1)+0.5*H
  EX(3)=ELX(2)
C
  DO 50 NI=1,NGP
  XI=GAUSPT(NI,NGP)
  CALL SHAPE1D(H,IEL,NPE,XI)
  CONST=GJ*GAUSWT(NI,NGP)
  X = 0.0
  DO 20 J=1,NPE
20  X = X + SF(J)*EX(J)
  C
  C   Compute the polynomial variation of FX
  C
  IF(NTYPE.EQ.2)THEN
    FX=FX0+(FX1+FX2*X)*X
  ELSE
    FX=(FX0+FX1*X)*X
  ENDIF
  C
  C   Element force vector for the consistent interpolation beam element
  C
25  II=1
  DO 40 I=1,NPE
  ELF(II)=ELF(II)+FX*SF(I)*CONST
40  II=NDF*I+1
50  CONTINUE
  C
  C   Rearrange the element coefficients
  C
  F3(NE)=ELF(3)
  ELF(1)=ELF(1)+0.5*F3(NE)
  ELF(2)=-0.125*F3(NE)*H
  ELF(3)=ELF(5)+0.5*F3(NE)
  ELF(4)= 0.125*F3(NE)*H
  RETURN
  END

```

```

SUBROUTINE TIMSTRES(GA,ELU,XI,W,DW,S,DS,NE,F3,H,MXELM)
-----
C
C   Called in POSTPROC to compute solution and its global derivatives
C   at nine points (including the nodes) of the Timoshenko element
C
C   XC..... Global (i.e., problem) coordinate
C   XI ..... Local (i.e., element) coordinate
C   SFL, SFQ.. Lagrange linear and quadratic shape functions
C   DSFL,DSFQ: First derivative of SF w.r.t. global coordinate

```

```

C      ELU..... Column vector of generalized displacements
C      W, DW..... Transverse deflection and its derivative
C      S, DS..... Rotation and its derivative
C      -----
C
C      IMPLICIT REAL*8 (A-H,O-Z)
C      COMMON/IO/IN,IT
C      DIMENSION ELU(9),SFL(2),SFQ(3),DSFL(2),DSFQ(3),F3(MXELM)
C
C      GJ = H*0.5
C
C      Interpolation functions for the Lagrange LINEAR element
C
C      SFL(1) = 0.5*(1.0-XI)
C      SFL(2) = 0.5*(1.0+XI)
C      DSFL(1) = -0.5/GJ
C      DSFL(2) = 0.5/GJ
C
C      Interpolation functions for the Lagrange QUADRATIC element
C
C      SFQ(1) = -0.5*XI*(1.0-XI)
C      SFQ(2) = 1.0-XI*XI
C      SFQ(3) = 0.5*XI*(1.0+XI)
C      DSFQ(1) = -0.5*(1.0-2.0*XI)/GJ
C      DSFQ(2) = -2.0*XI/GJ
C      DSFQ(3) = 0.5*(1.0+2.0*XI)/GJ
C
C      W3=(3.0*H*F3(NE)/GA + 8.0*(ELU(1)+ELU(3))
*      + 2.0*(ELU(4)-ELU(2))*H)/16.0
C      W = SFQ(1)*ELU(1) + SFQ(2)*W3 + SFQ(3)*ELU(3)
C      DW= DSFQ(1)*ELU(1) +DSFQ(2)*W3 +DSFQ(3)*ELU(3)
C      S = SFL(1)*ELU(2) + SFL(2)*ELU(4)
C      DS= DSFL(1)*ELU(2) +DSFL(2)*ELU(4)
C
C      RETURN
C      END
C
C      SUBROUTINE TRANSFRM(MXELM,N,NTYPE,PR,SE,SL,SA,SI,CS,SN,CNT,SNT,
*      HF,VF,PF,XB)
C      -----
C
C      Called in both MAIN and REACTION to compute stiffness matrix and
C      force vector for the truss (NDF=2) and frame (NDF=3) elements
C
C      SE.....Young's modulus
C      SL.....Element length
C      SA.....Cross-sectional area

```

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C      SI.....Moment of inertia
C      CS.....Cosine of the angle of orientation
C      SN.....Sine of the angle of orientation
C      HF.....Distributed force along the length of the element
C      VF.....Distributed force transverse to the element
C      PF.....Point force at point other than nodes
C      XB.....Distance along the length from node 1 of the element
C              of the location of the point force, PF
C      CNT,SNT:Direction cosines of the point force's line of application
C      -----
C
C      IMPLICIT REAL*8(A-H,O-Z)
C      DIMENSION PR(MXELM),SE(MXELM),SL(MXELM),SA(MXELM),SI(MXELM)
C      DIMENSION CS(MXELM),SN(MXELM),CNT(MXELM),SNT(MXELM)
C      DIMENSION HF(MXELM),VF(MXELM),PF(MXELM),XB(MXELM)
C      DIMENSION TRM(6,6),TMPK(6,6)
C      COMMON/STF1/ELK(9,9),ELM(9,9),ELF(9),ELX(4),ELU(9),ELV(9),ELA(9)
C
C      CN1=CS(N)
C      SN1=SN(N)
C      CN2=CN1*CN1
C      SN2=SN1*SN1
C      CSN=CN1*SN1
C
C      Element coefficients
C
C      IF(NTYPE.EQ.0) THEN
C
C      The plane TRUSS element
C
C      NN=4
C      C1=SA(N)*SE(N)/SL(N)
C      ELK(1,1) = C1*CN2
C      ELK(2,1) = C1*CSN
C      ELK(2,2) = C1*SN2
C      ELK(3,1) = -ELK(1,1)
C      ELK(3,2) = -ELK(2,1)
C      ELK(3,3) = ELK(1,1)
C      ELK(4,1) = -ELK(2,1)
C      ELK(4,2) = -ELK(2,2)
C      ELK(4,3) = -ELK(3,2)
C      ELK(4,4) = ELK(2,2)
C
C      DO 10 I=1,NN
C      DO 10 J=I,NN
10      ELK(I,J) = ELK(J,I)
C
C      Contribution of the point force to nodal forces

```

```

C
    XI=XB(N)/SL(N)
    SFL1 = 1.0-XI
    SFL2 = XI
C
    F1=0.5*HF(N)*SL(N)
    F3=0.5*HF(N)*SL(N)
    ELF(1) = F1*CN1
    ELF(2) = F1*SN1
    ELF(3) = F3*CN1
    ELF(4) = F3*SN1
ELSE
    NN=6
    IF(NTYPE.EQ.1) THEN
C
C
C      The EULER-BERNOULLI FRAME element
C
    AMU=0.5*SA(N)*SL(N)*SL(N)/SI(N)
    C1=2.0*SE(N)*SI(N)/(SL(N)**3)
    C2=6.0*SE(N)*SI(N)/(SL(N)*SL(N))
    C3=C1*(AMU*CN2+6.0*SN2)
    C4=C1*(AMU-6.0)*CSN
    C5=C1*(AMU*SN2+6.0*CN2)
    C6=4.0*SE(N)*SI(N)/SL(N)
C
    ELK(1,1) = C3
    ELK(2,1) = C4
    ELK(2,2) = C5
    ELK(3,1) = C2*SN1
    ELK(3,2) = -C2*CN1
    ELK(3,3) = C6
    ELK(4,1) = -C3
    ELK(4,2) = -C4
    ELK(4,3) = -C2*SN1
    ELK(4,4) = C3
    ELK(5,1) = -C4
    ELK(5,2) = -C5
    ELK(5,3) = C2*CN1
    ELK(5,4) = C4
    ELK(5,5) = C5
    ELK(6,1) = C2*SN1
    ELK(6,2) = -C2*CN1
    ELK(6,3) = 0.5*C6
    ELK(6,4) = -C2*SN1
    ELK(6,5) = C2*CN1
    ELK(6,6) = C6
C
    DO 20 I=1,NN
    DO 20 J=I,NN
20    ELK(I,J) = ELK(J,I)

```

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```

C
C   Contribution of the point force to nodal generalized forces
C
      XI=XB(N)/SL(N)
      TF=PF(N)*SNT(N)
      AF=PF(N)*CNT(N)
      SFL1 = 1.0-XI
      SFL2 = XI
      SFH1 = 1.0 - 3.0*XI*XI + 2.0*(XI**3)
      SFH2 = -XI*(1.0+XI*XI-2.0*XI)*SL(N)
      SFH3 = 3.0*XI*XI - 2.0*(XI**3)
      SFH4 = -XI*(XI*XI - XI)*SL(N)
C
      F1=0.5*HF(N)*SL(N)          + SFL1*AF
      F2=0.5*VF(N)*SL(N)          + SFH1*TF
      F3=-VF(N)*SL(N)*SL(N)/12.0 + SFH2*TF
      F4=0.5*HF(N)*SL(N)          + SFL2*AF
      F5=0.5*VF(N)*SL(N)          + SFH3*TF
      F6=VF(N)*SL(N)*SL(N)/12.0 + SFH4*TF
      ELF(1) = F1*CN1-F2*SN1
      ELF(2) = F1*SN1+F2*CN1
      ELF(3) = F3
      ELF(4) = F4*CN1-F5*SN1
      ELF(5) = F4*SN1+F5*CN1
      ELF(6) = F6
      ELSE
C
C   The TIMOSHENKO FRAME element (shear coefficient=5/6)
C
      SG=5.0*SE(N)/(1.0+PR(N))/12.0
      C1=SA(N)*SE(N)/SL(N)
      C2=SG*SA(N)/SL(N)
      C3=0.5*SG*SA(N)
      C4=0.25*SG*SA(N)*SL(N)
      C5=SE(N)*SI(N)/SL(N)
      ELK(1,1)=C1
      ELK(2,1)=0.0
      ELK(2,2)=C2
      ELK(3,1)=0.0
      ELK(3,2)=-C3
      ELK(3,3)=C4+C5
      ELK(4,1)=-C1
      ELK(4,2)=0.0
      ELK(4,3)=0.0
      ELK(4,4)=C1
      ELK(5,1)=0.0
      ELK(5,2)=-C2
      ELK(5,3)=C3
      ELK(5,4)=0.0
      ELK(5,5)=C2

```



```

        ELK(6,1)=0.0
        ELK(6,2)=-C3
        ELK(6,3)=C4-C5
        ELK(6,4)=0.0
        ELK(6,5)=C3
        ELK(6,6)=C4+C5
C
        DO 25 I=1,NN
        DO 25 J=1,NN
25      TRM(J,I)=0.0
C
        TRM(1,1)=CN1
        TRM(1,2)=SN1
        TRM(2,1)=-SN1
        TRM(2,2)=CN1
        TRM(3,3)=1.0
        TRM(4,4)=CN1
        TRM(4,5)=SN1
        TRM(5,4)=-SN1
        TRM(5,5)=CN1
        TRM(6,6)=1.0
C
        DO 30 I=1,NN
        DO 30 J=I,NN
30      ELK(I,J) = ELK(J,I)
C
        DO 40 I=1,NN
        DO 40 J=1,NN
        TMPK(I,J)=0.0
        DO 40 K=1,NN
40      TMPK(I,J)=TMPK(I,J)+TRM(K,I)*ELK(K,J)
C
        DO 50 I=1,NN
        DO 50 J=1,NN
        ELK(I,J)=0.0
        DO 50 K=1,NN
50      ELK(I,J)=ELK(I,J)+TMPK(I,K)*TRM(K,J)
C
C      Contribution of the point force to nodal generalized forces
C
        XI=XB(N)/SL(N)
        TF=PF(N)*SNT(N)
        AF=PF(N)*CNT(N)
        SFL1 = 1.0-XI
        SFL2 = XI
        SFQ1 = (1.0-XI)*(1.0-2.0*XI)
        SFQ2 = -XI*(1.0-2.0*XI)
        SFQ3 = 4.0*XI*(1.0-XI)
C

```

50 AN INTRODUCTION TO THE FINITE ELEMENT METHOD

```
F1=0.5*HF(N)*SL(N)          + SFL1*AF
F2=0.5*VF(N)*SL(N)          + (SFQ1+0.5*SFQ3)*TF
F3=-VF(N)*SL(N)*SL(N)/12.0 - 0.125*SFQ3*SL(N)*TF
F4=0.5*HF(N)*SL(N)          + SFL2*AF
F5=0.5*VF(N)*SL(N)          + (SFQ2+0.5*SFQ3)*TF
F6=VF(N)*SL(N)*SL(N)/12.0  + 0.125*SFQ3*SL(N)*TF
ELF(1) = F1*CN1-F2*SN1
ELF(2) = F1*SN1+F2*CN1
ELF(3) = F3
ELF(4) = F4*CN1-F5*SN1
ELF(5) = F4*SN1+F5*CN1
ELF(6) = F6
ENDIF
ENDIF
RETURN
END
```