**Chapter 11**

**11-1** Eq. (*b*), Sec. 11-3: *L*10 = 60 *LRnR* = 60(3000)500 = 90 (106) rev

Eq. (11-3):



\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_

**11-2** For the deep-groove 02-series ball bearing with *R* = 0.90, the design life *xD*, in multiples of rating life, is



The design radial load is



Eq. (11-9): 

*C*10 = 24.3 kN *Ans.*

Table 11-2: Choose an 02-35 mm bearing with *C*10 = 25.5 kN. *Ans.*

Eq. (11-21): 

**\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_**

**11-3** For the angular-contact 02-series ball bearing as described, the rating life multiple is



The design radial load is



Eq. (11-9):



Table 11-2: Select an02-60 mm bearing with *C*10 = 55.9 kN. *Ans.*

Eq. (11-21): 

**\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_**

**11-4** For the straight-roller 03-series bearing selection, *xD* = 1248 rating lives from Prob. 11-3 solution.





Table 11-3: Select an 03-60 mm bearing with *C*10 = 123 kN. *Ans.*

Eq. (11-21): 

**\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_**

**11-5** The combined reliability of the two bearings selected in Probs. 11-3 and 11-4 is



We can choose a reliability goal of for each bearing. We make the selections, find the existing reliabilities, multiply them together, and observe that the reliability goal is exceeded due to the roundup of capacity upon table entry.

Another possibility is to use the reliability of one bearing, say *R*1. Then set the reliability goal of the second as



or vice versa. This gives three pairs of selections to compare in terms of cost, geometry implications, etc.

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**11-6** Establish a reliability goal of for each bearing. For an 02-series angular contact ball bearing,



Select an 02-65 mm angular-contact bearing with *C*10 = 63.7 kN.



For an 03-series straight roller bearing,



Select an 03-65 mm straight-roller bearing with *C*10 = 138 kN.



The overall reliability is *R* = (0.962)(0.953) = 0.917, which exceeds the goal.

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**11-7** Given: *R* = 96 percent, *αf* = 1.2, *x*0 = 0, *θ* = 4.48, *b* = 1.5. From Prob. 11-1,

*FD* = 2.75 kN, *LR* = *L*10 = 60 *LRnR* = 60(3000)500 = 90 (106) rev, and

*LD* = 60 *LDnD* = 60(10)103(1800) = 1.08 (109) rev

The dimensionless multiple of rating life is



Eq. (11-10):



\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_

**11-8** For the straight cylindrical roller bearing specified with a service factor of 1, *R* = 0.95 and *FR* = 20 kN.





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**11-9** Both bearings need to be rated in terms of the same catalog rating system in order to compare them. Using a rating life of one million revolutions, both bearings can be rated in terms of a Basic Load Rating.

Eq. (11-3): 

Bearing *B* already is rated at one million revolutions, so *CB* = 7.0 kN. Since *CA*>*CB*, bearing *A* can carry the larger load. *Ans.*

\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_

**11-10** *FD* = 2 kN, *LD* = 109 rev, *R* = 0.90

Eq. (11-3): 

\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_

**11-11** *FD* = 3248 N, *D* = 12 000 hours, *nD* = 350 rev/min, *R* = 0.90

Eq. (11-3): 

\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_

**11-12** *FD* = 4 kN, *D* = 8000 hours, *nD* = 500 rev/min, *R* = 0.90

Eq. (11-3): 

\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_

**11-13** *FD* = 2912 N, *nD* = 400 rev/min, *R* = 0.95**

*D* = (5 years)(40 h/week)(52 week/year) = 10 400 hours

Assume an application factor of one.The multiple of rating life is



Eq. (11-9): 



\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_

**11-14** *FD* = 9 kN, *LD* = 108 rev, *R* = 0.99

Assume an application factor of one.The multiple of rating life is



Eq. (11-9): 



\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_

**11-15** *FD* = 49280 N, *D* = 20 000 hours, *nD* = 200 rev/min, *R* = 0.99

Assume an application factor of one. Use the Weibull parameters for Manufacturer 2 in Table 11-6.

The multiple of rating life is



Eq. (11-9): 



\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_

**11-16** From the solution to Prob. 3-79, the ground reaction force carried by the bearing at *C* is *RC* =*FD* = 792.86 N. Use the Weibull parameters for Manufacturer 2 in Table 11-6.



Eq. (11-10): 



\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_

**11-17** From the solution to Prob. 3-80, the ground reaction force carried by the bearing at *C* is *RC* = *FD* = 1.794 kN. Use the Weibull parameters for Manufacturer 2 in Table 11-6.



Eq. (11-10): 



\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_

**11-18** From the solution to Prob. 3-81, *RCz* = –1458.9 N, *RCy* = –566.10 N



Use the Weibull parameters for Manufacturer 2 in Table 11-6.



Eq. (11-10): 



\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_

**11-19** From the solution to Prob. 3-82, *RCz* = –150.7 N, *RCy* = –86.10 N



Use the Weibull parameters for Manufacturer 2 in Table 11-6.



Eq. (11-10): 



\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_

**11-20** From the solution to Prob. 3-88, *RAz* = 444 N, *RAy* = 2384 N



Use the Weibull parameters for Manufacturer 2 in Table 11-6.The design speed is equal to the speed of shaft *AD*,





Eq. (11-10): 



\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_

**11-21** From the solution to Prob. 3-90, *RAz* = 240.2 N, *RAy* = 622.72 N 

Use the Weibull parameters for Manufacturer 2 in Table 11-6.The design speed is equal to the speed of shaft *AD*,





Eq. (11-10): 



\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_

**11-22 (a)** 

From Table 11-2, with a 65 mm bore, *C*0 = 34.0 kN.

*Fa* / *C*0 = 3 / 34 = 0.088

From Table 11-1, 0.28 ≤*e*≤ 3.0.



Since this is greater than *e*, interpolating Table 11-1 with *Fa* / *C*0 = 0.088, we obtain

*X*2= 0.56 and *Y*2 = 1.53.

Eq. (11-12):  *Ans.*

*Fe*>*Fr* so use *Fe*.

**(b)** Use Eq. (11-10) to determine the necessary rated load the bearing should have to carry the equivalent radial load for the desired life and reliability. Use the Weibull parameters for Manufacturer 2 in Table 11-6.



Eq. (11-10): 



From Table 11-2, the 65 mm bearing is rated for 55.9 kN, which is less than the necessary rating to meet the specifications. This bearing should not be expected to meet the load, life, and reliability goals. *Ans.*

\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_

**11-23 (a)** 

From Table 11-2, 30 mm bore, *C*10 = 19.5 kN, *C*0 = 10.0 kN

*Fa* / *C*0 = 2 / 10 = 0.2

From Table 11-1, 0.34≤*e*≤0.38.



Since this is greater than *e*, interpolating Table 11-1, with *Fa* / *C*0 = 0.2, we obtain *X*2= 0.56 and *Y*2 = 1.27.

Eq. (11-12): *Ans.*

*Fe*>*Fr* so use *Fe*.

**(b)** Solve Eq. (11-10) for *xD*.











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**11-24 **

Eq. (11-3): 

From Table 11-2, select the 85 mm bore. *Ans.*

\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_

**11-25 **

Use the Weibull parameters for Manufacturer 2 in Table 11-6.



First guess: Choose from middle of Table 11-1, *X* = 0.56, *Y* = 1.63

Eq. (11-12): 

*Fe*<*Fr*, so just use *Fr­* as the design load.

Eq. (11-10): 



From Table 11-2, try 85 mm bore with *C*10 = 83.2 kN, *C*0 = 53.0 kN

Iterate the previous process:

*Fa* / *C*0 = 2 / 53 = 0.038

Table 11-1: 0.22 ≤*e*≤ 0.24



Interpolate Table 11-1 with *Fa* / *C*0 = 0.038 to obtain *X*2 = 0.56 and *Y*2 = 1.89.

Eq. (11-12): 

Eq. (11-10): 

Table 11-2: Move up to the 90 mm bore with *C*10 = 95.6 kN, *C*0 = 62.0 kN.

Iterate again:

*Fa* / *C*0 = 2 / 62 = 0.032

Table 11-1: Again, 0.22 ≤*e*≤ 0.24



Interpolate Table 11-1 with *Fa* / *C*0 = 0.032 to obtain *X*2 = 0.56 and *Y*2 = 1.95.

Eq. (11-12): 

Eq. (11-10): 

The 90 mm bore is acceptable. *Ans.*

\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_

**11-26 **

First guess: Choose from middle of Table 11-1, *X* = 0.56, *Y* = 1.63

Eq. (11-12): 



Eq. (11-3): 

From Table 11-2, try 60 mm with *C*10 = 47.5 kN, *C*0 = 28.0 kN

Iterate the previous process:

*Fa* / *C*0 = 3 / 28 = 0.107

Table 11-1: 0.28 ≤*e*≤ 0.30



Interpolate Table 11-1 with *Fa* / *C*0 = 0.107 to obtain *X*2 = 0.56 and *Y*2 = 1.46

Eq. (11-12): 

Eq. (11-3): 

From Table 11-2, we have converged on the 60 mm bearing.*Ans.*

\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_

**11-27 **

Use the Weibull parameters for Manufacturer 2 in Table 11-6.



First guess: Choose from middle of Table 11-1, *X* = 0.56, *Y* = 1.63

Eq. (11-12): 

*Fe*>*Fr*, so use *Fe­* as the design load.

Eq. (11-10): 



From Table 11-2, try 95 mm bore with *C*10 = 108 kN, *C*0 = 69.5 kN

Iterate the previous process:

*Fa* / *C*0 = 5 / 69.5 = 0.072

Table 11-1: 0.27 ≤*e*≤ 0.28



Interpolate Table 11-1 with *Fa* / *C*0 = 0.072 to obtain *X*2 = 0.56 and *Y*2 = 1.62 1.63

Since this is where we started, we will converge back to the same bearing. The 95 mm bore meets the requirements. *Ans.*

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**11-28 **

Use the Weibull parameters for Manufacturer 2 in Table 11-6.



First guess: Choose from middle of Table 11-1, *X* = 0.56, *Y* = 1.63

Eq. (11-12): 

*Fe*>*Fr*, so use *Fe­* as the design load.

Eq. (11-10): 



From Table 11-2, try 90 mm bore with*C*10 = 95.6 kN, *C*0 = 62.0 kN. Try this bearing.

Iterate the previous process:

*Fa* / *C*0 = 3 / 62 = 0.048

Table 11-1: 0.24 ≤*e*≤ 0.26



Interpolate Table 11-1 with *Fa* / *C*0 = 0.048 to obtain *X*2 = 0.56 and *Y*2 = 1.79

Eq. (11-12): 



From Table 11-2, this converges back to the same bearing. The 90 mm bore meets the requirements.*Ans.*

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**11-29** **(a)** 

From Prob. 3-83, *RCy* = 814.43 N, *RCz* = –3831.95 N.





Eq. (11-10): 



**(b)** Results will vary depending on the specific bearing manufacturer selected. A general engineering components search site such as www.globalspec.com might be useful as a starting point.

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**11-30** **(a)** 

From Prob. 3-83, *ROy* = –927.41 N, *ROz* = 1153.37 N.





Eq. (11-10): 



**(b)** Results will vary depending on the specific bearing manufacturer selected. A general engineering components search site such as www.globalspec.com might be useful as a starting point.

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**11-31** **(a)** 

From Prob. 3-84, *RCy* = 8.319 kN, *RCz* = –10.830 kN.





Eq. (11-10):

**(b)** Results will vary depending on the specific bearing manufacturer selected. A general engineering components search site such as www.globalspec.com might be useful as a starting point.

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**11-32 (a)** 

From Prob. 3-84, *ROy* = 5083 N, *ROz* = 494 N.

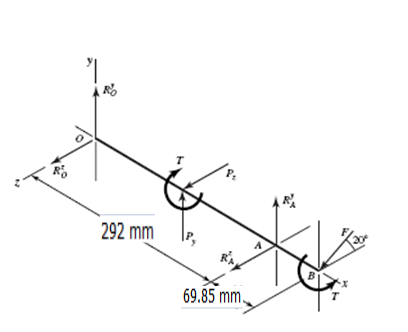




Eq. (11-10):

**(b)** Results will vary depending on the specific bearing manufacturer selected. A general engineering components search site such as www.globalspec.com might be useful as a starting point.

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**11-33** Assume concentrated forces as shown.





































So the reaction at *A* governs.

Reliability Goal: 



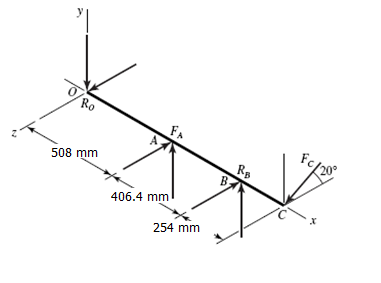




From Table 11-2, a 40 mm boreangular contact bearing is sufficient with a rating of

31.9 kN. *Ans.*

\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_

**11-34** For a combined reliability goal of 0.95, usefor the individual bearings. 



The resultants of the given forces are

*RO* = [(–1721.38)2 + 2077.222]1/2 = 2700 N

*RB* = [1405.572 + (–7183.52)2]1/2 = 7321.41 N

At *O*:

Eq. (11-9):  

From Table 11-2, select an 02-55 mm angular-contact ball bearing with a basic load rating of 46.2 kN. *Ans.*

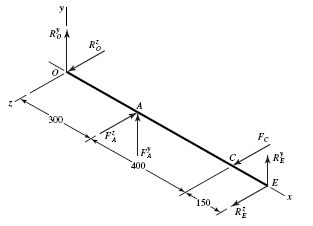
At *B*:

Eq. (11-9):  

From Table 11-3, select an 02-75 mm or 03-55 mm cylindrical roller. *Ans.*

\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_

**11-35** The reliability of the individual bearings is 

 From statics,

*T* = (270 − 50) = (*P*1−*P*2)125

= (*P*1− 0.15 *P*1)125

*P*1 = 310.6 N,

*P*2 = 0.15 (310.6) = 46.6 N

*P*1 +*P*2 = 357.2 N 











The radial loads are nearly the same at *O* and *E*. We can use the same bearing at both locations.

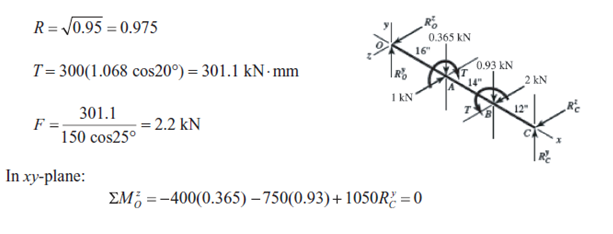


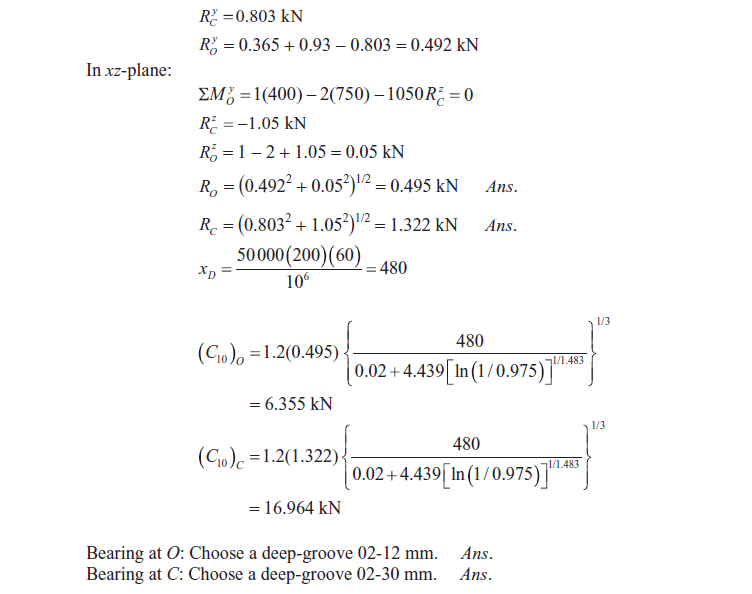
Eq. (11-9): 

From Table 11-2, select an 02-12 mm deep-groove ball bearing with a basic load rating of 6.89 kN.*Ans.*

\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_

**11-36**





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**11-37** Shafts subjected to thrust can be constrained by bearings, one of which supports the thrust. The shaft floats within the endplay of the second (roller) bearing. Since the thrust force here is larger than any radial load, the bearing absorbing the thrust (bearing *A*) is heavily loaded compared to bearing*B*. Bearing *B* is thus likely to be oversized and may not contribute measurably to the chance of failure. If this is the case, we may be able to obtain the desired combined reliability with bearing *A* having a reliability near 0.99 and bearing *B* having a reliability near 1. This would allow for bearing *A* to have a lower capacity than if it needed to achieve a reliability of . To determine if this is the case, we will start with bearing *B*.

*Bearing B* (straight roller bearing)





Try a reliability of 1 to see if it is readily obtainable with the available bearings.

Eq. (11-9): 

The smallest capacity bearing from Table 11-3 has a rated capacity of 16.8 kN. Therefore, we select the 02-25 mm straight cylindrical roller bearing. *Ans.*

*Bearing at A* (angular-contact ball)

With a reliability of 1 for bearing *B*, we can achieve the combined reliability goal of 0.99 if bearing *A* has a reliability of 0.99.





Trial #1:

Tentatively select an 02-85 mm angular-contact with *C*10 = 90.4 kN and *C*0 = 63.0 kN.





Table 11-1: Interpolating, *X*2 = 0.56, *Y*2 = 1.88

Eq. (11-12): 

Eq. (11-9): 



Trial #2:

Tentatively select a 02-90 mm angular-contact ball with *C*10 = 106 kN and *C*0 = 73.5 kN.



Table 11-1: Interpolating, *X*2 = 0.56, *Y*2 = 1.93





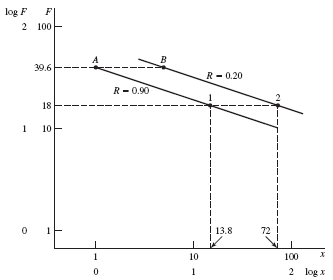
Select an 02-90 mm angular-contact ball bearing. *Ans.*

\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_

**11-38** We have some data. Let’s estimate parameters *b* and *θ* from it. In Fig. 11-5, we will use line *AB*. In this case, *B* is to the right of *A*.

For *F* = 18 kN, 

This establishes point 1 on the *R* = 0.90 line.



The *R* = 0.20 locus is above and parallel to the *R* = 0.90 locus. For the two-parameter Weibull distribution, *x*0 = 0 and points *A* and *B* are related by [see Eq. (11-8)]:

 (1)



and *xB*/*xA* is in the same ratio as 600/115. Eliminating *θ*,



Solving for *θ* in Eq. (1),



Therefore, for the data at hand,



Check *R* at point *B*: *xB* = (600/115) = 5.217



Note also, for point 2 on the *R* = 0.20 line,





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**11-39** This problem is rich in useful variations. Here is one.

*Decision*: Make straight roller bearings identical on a given shaft. Use a reliability goal of (0.99)1/6 = 0.9983.

*Shaft a*





Thus the bearing at *B* controls.







Select an 02-80 mm with *C*10 = 106 kN.*Ans.*

*Shaft b*





The bearing at *C* controls.





Select an 02-90 mm with *C*10 = 142 kN.*Ans.*

*Shaft c*





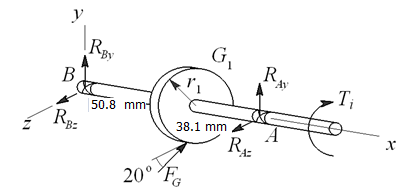
The bearing at *E* controls.





Select an 02-80 mm with *C*10 = 106 kN.*Ans.*

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**11-40** **(a)** Eq. (1-9): 

**(b)**

Σ*Mx* = 0 = (*FG* cos 20o)(1) – 889.6

*FG* = 946.71 N

Σ(*MB*)*y* = 0 = 50.8(*FG* cos 20o) – 88.9*RAz*

*RAz* = 50.8(946.71 cos 20o)/88.9 = 508.36 N

Σ(*MB*)*z* = 0, 50.8 (946.71 sin 20o) + 88.9 (*FA*)*y* = 0 ⇒*RAy* = *–* 185.04 N



**(c)***L*10 = 1 (106) revolutions, *LD* = 30(103)60(60) = 108(106) revolutions.



Eq. (11-10) with *af* = 1, *xD* = 108, *a* = 3, *FD* = 540.88 N, *RD* = 0.9794, *x*0 = 0.02,

*θ* = 4.459, and *b* =1.483:



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**11-41** For the output shaft: *N*1 /*N*2 = *ω*2 /*ω*1⇒*ω*2 = (*N*1 /*N*2)*ω*2 = (30/60) 200 = 100 rev/min.

Service time in minutes: 

For bearings *C* and *D*: *LD* = 748.8(103)100 = 74.88(106) revolutions



**(a)***R* = *RARBRCRD* ⇒*RD* = *R* / (*RARBRC*) = 0.90 / [0.97(0.96)0.98] = 0.986 *Ans*.

**(b)** Given *Fr* = 9 kN, *Fa* = 5 kN. Eq. (11-12) with *V* = 1 (inner ring rotates):

*Fe* = *Xi Fr* + *YiFa*

Iteration will be necessary. First try. From the middle of Table 11-1, *X*2 = 0.56, *Y*2 = 1.63

*Fe* = 0.56(9) + 1.63(5) = 13.19 kN

Eq. (11-10):

From Table 11-2 we see that *C*10 exceeds the 83.2 kN rating for an 85 mm bore bearing. To see if this bearing is possible, we need to determine *Xi*and *Yi*. For the 85 mm bearing:

*C*0 = 53.0 kN, *Fa* / *C*0 = 5/53 = 0.0943

*Fa* / *C*0 *e*

0084 0.28

0.0943 *e*

0.110 0.30



. Since 0.556 > 0.29, use *X*2 = 0.56, *Y*2 = 1.50 (interpolated)

*Fe* = 0.56(9) + 1.50(5) = 12.54 kN



Since 81.8 kN < 83.2 kN it is acceptable to use the bearing with the 85 mm bore. *Ans*.

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**11-42** From Table 11-3, for a 03-30 mm cylindrical roller bearing,

*C*10 =36.9 kN = 36.9/4.45 = 8.292 kips.

Eq. (*a*), Sec. 11-7: 

At 5 kips: 

At 8 kips: 

Eq. (11-16): 



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**11-43** From Table 11-3, select a 25 mm trial bearing. Using the equations and 

Size *C*10  *L*1 *L*2 *L*3­­­­­­­­­­­­­­­­­­­­­­­

25 mm 28.6 kN    

7.154(1010) 3.294(106) 1.566(106) 852.7(103)



Using a spreadsheet following the method above yields

Size *l*

25 mm 1.55(106) rev

30 mm 3.63(106) rev

35 mm 6.82(106) rev greater than 5 (106) rev. Thus, 35 mm *Ans*.

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**11-44** Express Eq. (11-1) as



For a ball bearing, *a* = 3 and for an 02-30 mm angular contact bearing, *C*10 = 20.3 kN.



At a load of 18 kN, life *L*1 is given by:



For a load of 30 kN, life *L*2 is:



In this case, Eq. (6-68) – the Palmgren-Miner cycle-ratio summation rule – can be expressed as



Substituting,





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**11-45** *Total life in revolutions*

Let:

*l* = total turns

*f*1 = fraction of turns at *F*1

*f2* = fraction of turns at *F*2

From the solution of Prob. 11-44, *L*1 = 1.434(106) rev and *L*2 = 0.310(106) rev.

Palmgren-Miner rule:



from which





*Total life in loading cycles*

4 min at 2000 rev/min = 8000 rev/cycle

6 min at 2000 rev/min = 12 000 rev/cycle

Total rev/cycle = 8000 + 12 000 = 20 000



*Total life in hours*



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**11-46**  









Eq. (11-18): 

Eq. (11-18): 





We will size bearing *B* first since its induced load will affect bearing *A*, but is not itself affected by the induced load from bearing *A* [see Eq. (11-19)].

From Eq. (11-19*b*), *FeB* = *FrB* = 4870.56 N

Eq. (11-10):  *Ans.*

Select cone 32305, cup 32305, with 25 mm bore, and rated at 17.39 kN with

*K* = 1.95.*Ans.*

With bearing *B* selected, we re-evaluate the induced load from bearing *B* using the actual value for *K.*

Eq. (11-18): 

Find the equivalent radial load for bearing *A* from Eq. (11-19), which still applies.

Eq. (11-19*a*): 





Eq. (11-10): 

Tentatively select cone M86643, cup M86610, with 25.4 mm bore, and rated at 14.456 kN

with *K* = 1.07. Iterating with the new value for *K*, we get *FeA*= 3.122 kN and *FrA* = 10.283 kN.*Ans.*

By using a bearing with a lower *K*, the rated load decreased significantly, providing a higher than requested reliability. Further examination with different combinations of bearing choices could yield additional acceptable solutions.

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**11-47** The thrust load on shaft *CD* is from the axial component of the force transmitted through the bevel gear, and is directed toward bearing *C*. By observation of Fig. 11-14, direct mounted bearings would allow bearing *C* to carry the thrust load. *Ans.*

From the solution to Prob. 3-85, the axial thrust load is *Fae* = 1613.73 N, and the bearing radial forces are*FCx* = 1277.47 N, *FCz* = 2228 N, *FDx* = 864.69 N, and*FDz* = 1365.98 N. Thus, the radial forces are





The induced loads are

Eq. (11-18): ****

Eq. (11-18): ****

Check the condition on whether to apply Eq. (11-19) or Eq. (11-20), where bearings *C* and *D* are substituted, respectively, for labels *A* and *B* in the equations.

****

****

Eq. (11-19*a*): ****

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Assume for tapered roller bearings that the specifications for Manufacturer 25.4 mm Table 11-6 are applicable.





Eq. (11-10): 

Eq. (11-19*b*): 

Eq. (11-10): 

\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_

**11-48** The thrust load on shaft *AB* is from the axial component of the force transmitted through the bevel gear, and is directed to the right. By observation of Fig. 11-14, indirect mounted bearings would allow bearing *A* to carry the thrust load. *Ans.*

From the solution to Prob. 3-87, the axial thrust load is *Fae* = 412.77 N, and the bearing radial forces are *FAy* = 2844.05 N, *FAz* = 6732.94 N, *FBy* = 1230.32 N, and *FBz* = 3138.95 N. Thus, the radial forces are





The induced loads are9

Eq. (11-18): ****

Eq. (11-18): ****

Check the condition on whether to apply Eq. (11-19) or Eq. (11-20).

****

****

Notice that the induced load from bearing *A* is sufficiently large to cause a net axial force to the left, which must be supported by bearing *B.*

Eq. (11-20*a*): ****

****

Assume for tapered roller bearings that the specifications for Manufacturer 25.4 mm Table 11-6 are applicable.





Eq. (11-10): 

Eq. (11-19*b*): 

Eq. (11-10): 

\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_

**11-49** The lower bearing is compressed by the axial load, so it is designated as bearing *A*.

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Eq. (11-18): ****

Eq. (11-18): ****

Check the condition on whether to apply Eq. (11-19) or Eq. (11-20)

****

****

Eq. (11-19*a*): ****

****

****

Assume for tapered roller bearings that the specifications for Manufacturer 25.4 mm Table 11-6 are applicable.

Eq. (11-3): ****

Eq. (11-19*b*): 

Eq. (11-3): 

\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_

**11-50** The left bearing is compressed by the axial load, so it is properly designated as bearing *A*.

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Assume *K* = 1.5 for each bearingfor the first iteration. Obtain the induced loads.

Eq. (11-18): ****

Eq. (11-18): ****

Check the condition on whether to apply Eq. (11-19) or Eq. (11-20).

****

****

We will size bearing *B* first since its induced load will affect bearing *A*, but it is not affected by the induced load from bearing *A* [see Eq. (11-19)].

From Eq. (11-19*b*), *FeB* = *FrB* = 2800 N.

Eq. (11-3): ****

****

Select cone 07100, cup 07196, with 25.4 mm bore, and rated at 6983.36 with *K* = 1.45. *Ans.*

With bearing *B* selected, we re-evaluate the induced load from bearing *B* using the actual value for *K.*

Eq. (11-18): 

Find the equivalent radial load for bearing *A* from Eq. (11-19), which still applies.

Eq. (11-19*a*): 





Eq. (11-3): ****

****

Any of the bearings with 25.4-3.175 mm bore are more than adequate. Select cone 15590, cup 15520, rated at 11031 N with *K* = 1.69. Iterating with the new value for *K*, we get *FeA*= 4981.76 N and *FrA* = 9607.68 N. The selected bearing is still adequate. *Ans.*

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